# FORMULAS IN GEARING



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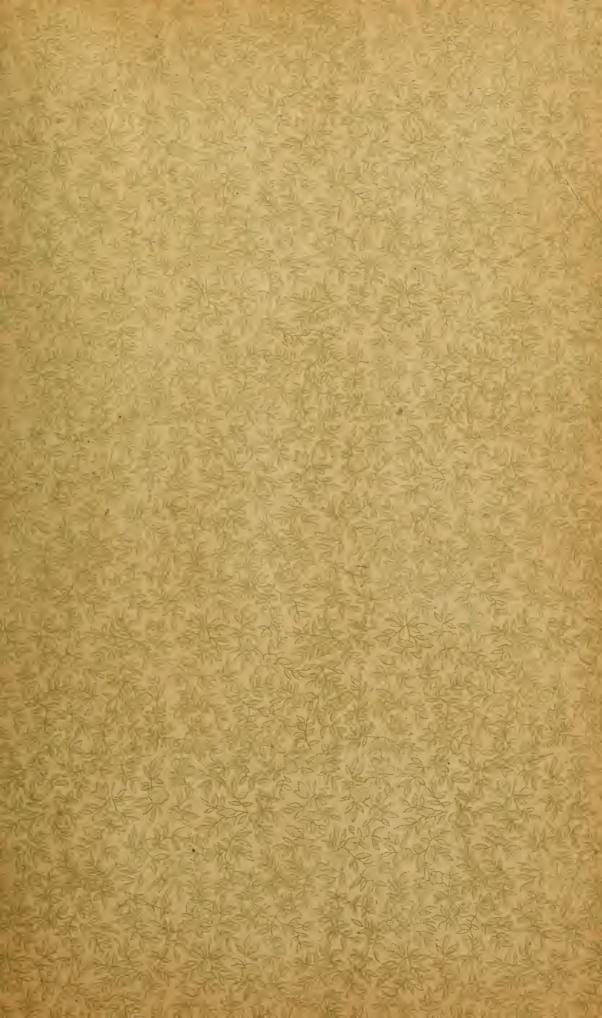
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## FORMULAS

IN

# GEARING.

THIRD EDITION.

WITH PRACTICAL SUGGESTIONS.

PROVIDENCE, R. I.

BROWN & SHARPE MANUFACTURING COMPANY

1900.

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## PREFACE.

It is the aim, in the following pages, to condense as much as possible the solution of all problems in gearing which in the ordinary practice may be met with, to the exclusion of problems dealing with transmission of power and strength of gearing. The simplest and briefest being the symbolical expression, it has, whenever available, been resorted to. The mathematics employed are of a simple kind, and will present no difficulty to anyone familiar with ordinary Algebra and the elements of Trigonometry.



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## FORMULAS IN GEARING.

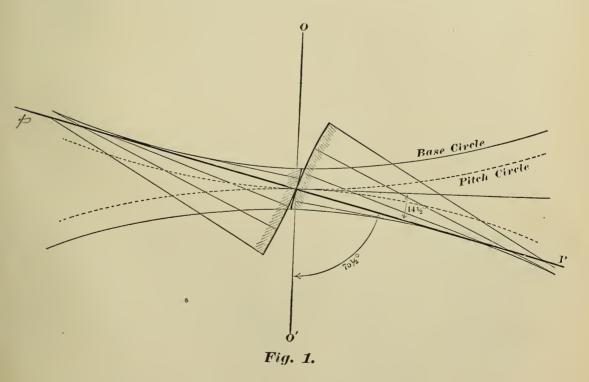
#### CHAPTER I.

### SYSTEMS OF GEARING.

(Figs. 1, 2.)

There are in common use two systems of gearing, viz.: the involute and the epicycloidal.

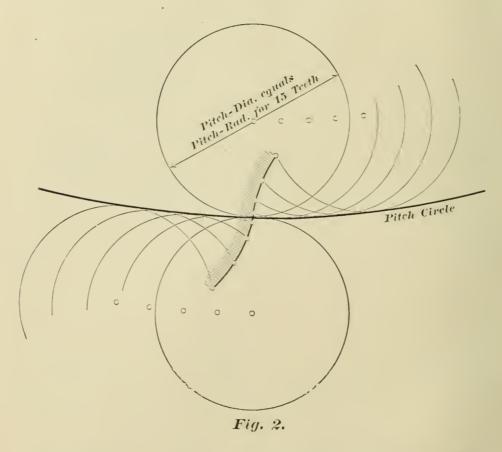
In the involute system the outlines of the working parts of a tooth are single curves, which may be traced by a point in a flexible, inextensible cord being unwound from a circular disk the circumference of which is called the base circle, the disk being concentric with the pitch circle of the gear.



In Fig. 1 the two base circles are represented as tangent to the line PP. This line (PP) is variously called "the line of pressure," "the line of contact," or "the line of action." In our practice this is drawn so as to make with a normal to the center line (O O')  $14\frac{1}{2}^{\circ}$ , or with the center line  $75\frac{1}{2}^{\circ}$ .

The rack of this system has teeth with straight sides, the two sides of a tooth making, together, an angle of  $29^{\circ}$  (twice  $14\frac{1}{2}^{\circ}$ ).

This applies to gears having 30 teeth or more. For gears having less than 30 teeth special rules are followed, which are explained in our "Practical Treatise on Gearing."



In epicycloidal, or double-curve teeth, the formation of the curve changes at the pitch circle. The outline of the faces of epicycloidal teeth may be traced by a point in a circle rolling on the outside of pitch circle of a gear, and the flanks by a point in a circle rolling on the inside of the pitch circle. The faces of one gear must be traced by the same circle that traces the flanks of the engaging gear.

In our practice the diameter of the rolling or describing circle is equal to the radius of a 15-tooth gear of the pitch required; this is the base of the system. The same describing circle being used for all gears of the same pitch.

The teeth of the rack of this system have double curves, which may be traced by the base circle rolling alternately on each side of the pitch line.

An advantage of the involute over the epicycloidal tooth is, that in action gears having involute teeth may be separated a little from their normal positions without interfering with the angular velocity, which is not possible in any other kind of tooth.

The obliquity of action is sometimes urged as an objection to involute teeth, but a full consideration of the subject will show that the importance of this has been greatly over-estimated.

The tooth dimensions for both the involute and epicycloidal gears may be calculated from the formulas in Chapter II.

#### CHAPTER II.

### SPUR GEARING.

(Figs. 3, 4.)

Two spur gears in action are comparable to two corresponding plain rollers whose surfaces are in contact, these surfaces representing the pitch circles of the gears.

#### PITCH OF GEARS.

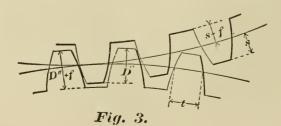
For convenience of expression the pitch of gears may be stated as follows:

Circular pitch is the distance from the center of one tooth to the center of the next tooth, measured on the pitch line.

Diametral pitch is the number of teeth in a gear per inch of pitch diameter. That is, a gear that has, say, six teeth for each inch in pitch diameter is six diametral pitch, or, as the expression is universally abbreviated, it is "six pitch." This is by far the most convenient way of expressing the relation of diameter to number of teeth.

Module is the pitch diameter of a gear divided by the number of teeth.

Chordal pitch is a term but little employed. It is the distance from center to center of two adjacent teeth measured in a straight line.



#### FORMULAS.

N = number of teeth.

s = addendum and module.

t =thickness of tooth on pitch line.

f = clearance at bottom of tooth.

D'' =working depth of tooth.

D'' + f =whole depth of tooth.

d = pitch diameter.

d' = outside diameter.

P' = circular pitch.

 $P^c$  = chord pitch.

P = diametral pitch.

C = center distance.

$$P = \frac{N + 2}{d'}$$

$$P = \frac{\pi}{P'}$$

$$P' = \frac{\pi}{P}$$

$$s = \frac{I}{P} = \frac{P'}{\pi} = .3183 P'$$

$$s = \frac{d}{N} = \frac{d'}{N + 2}$$

$$t = \frac{I}{2} P' = \frac{\pi}{2 P}$$

$$f = \frac{I}{10} t$$

$$s + f = \frac{I}{P} \left( I + \frac{\pi}{20} \right) = .3685 P'$$

$$D'' = 2 s$$

$$P^{c} = d \sin \frac{180^{\circ}}{N}$$

$$P' = d \pi \frac{\delta}{360^{\circ}} \text{ where } \sin \delta = \frac{P^{c}}{d}$$

$$d = \frac{N}{P}$$

$$d' = d + 2 s$$

$$d = \frac{N P'}{\pi}$$

GEAR WHEELS.

TABLE OF TOOTH PARTS—CIRCULAR PITCH IN FIRST COLUMN.

Circular Pitch.	Threads or Tecth per inch Linear.	Diametral Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.	Width of Thread-Tool at End.	Width of Thread at Top.
P'	1" P'	P	t	s	D"	s+f	D''+f	PX.31	P'X.335
2	1/2	1.5708	1.0000	.6366	1.2732	.7366	1.3732	.6200	.6700
$1\frac{7}{8}$	<u>8</u> 15	1.6755	.9375	.5968	1.1937	.6906	1.2874	.5813	.6281
$1\frac{3}{4}$	4 7	1.7952	.8750	.5570	1.1141	.6445	1.2016	.5425	.5863
$1\frac{5}{8}$	8 13	1.9333	.8125	.5173	1.0345	.5985	1.1158	.5038	.5444
$1\frac{1}{2}$	3	2.0944	.7500	.4775	.9549	.5525	1.0299	.4650	.5025
$1\frac{7}{16}$	$\frac{16}{23}$	2.1855	.7187	.4576	.9151	.5294	.9870	.4456	.4816
$1\frac{3}{8}$	<u>8</u> 11	2.2848	.6875	.4377	.8754	.5064	.9441	.4262	.4606
$1\frac{1}{3}$	3 4	2.3562	.6666	.4244	.8488	.4910	.9154	.4133	.4466
$1_{\frac{5}{16}}$	16 21	2.3936	.6562	.4178	.8356	.4834	.9012	.4069	.4397
$1\frac{1}{4}$	5	2.5133	.6250	.3979	.7958	.4604	.8583	.3875	.4188
$1\frac{3}{16}$	16 19	2.6456	.5937	.3780	.7560	.4374	.8156	.3681	.3978
$1\frac{1}{8}$	8 9	2.7925	.5625	.3581	.7162	.4143	.7724	.3488	.3769
$1\frac{1}{16}$	16 17	2.9568	.5312	.3382	.6764	.3913	.7295	.3294	.3559
1	1	3.1416	.5000	.3183	.6366	.3683	.6866	.3100	.3350
$\frac{15}{16}$	$1\frac{1}{15}$	3.3510	.4687	.2984	.5968	.3453	.6437	.2906	.3141
7 8	$1\frac{1}{7}$	3.5904	.4375	.2785	.5570	.3223	.6007	.2713	.2931
.13	$1\frac{3}{13}$	3.8666	.4062	.2586	.5173	.2993	.5579	.2519	.2722
<u>4'</u> 5	$1\frac{1}{4}$	3.9270	.4000	.2546	.5092	.2946	.5492	.2480	.2680
3 4	$1\frac{1}{3}$	4.1888	.3750	.2387	.4775	.2762	.5150	.2325	.2513
11/16	$1\frac{5}{11}$	4.5696	.3437	.2189	.4377	.2532	.4720	.2131	.2303
$\frac{2}{3}$	$1\frac{1}{2}$	4.7124	.3333	.2122	.4244	.2455	.4577	.2066	.2233
<u>5</u> 8	$1\frac{3}{5}$	5.0265	.3125	.1989	.3979	.2301	.4291	.1938	.2094
3 5	$1^{\frac{2}{3}}$	5.2360	.3000	.1910	.3820	.2210	.4120	.1860	.2010
4 7	$1\frac{3}{4}$	5.4978	.2857	.1819	.3638	.2105	.3923	.1771	.1914
9 16	$1\frac{7}{9}$	5.5851	.2812	.1790	.3581	.2071	.3862	.1744	.1884

### TABLE OF TOOTH PARTS.—Continued.

CIRCULAR PITCH IN FIRST COLUMN.

Circular Pitch.	Threads or Teeth per inch Linear.	Diametral Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.	Width of Thread-Tool at End.	Width of Thread at Top.
P'	1" P'	P	t	8 3	D''	s+f	D''+f.	P×.31	P×.335
1 2	2	6.2832	.2500	.1592	.3183	.1842	.3433	.1550	.1675
<u>0</u>	$2\frac{1}{4}$	7.0685	.2222	.1415	.2830	.1637	.3052	.1378	.1489
7	$2\frac{2}{7}$	7.1808	.2187	.1393	.2785	.1611	.3003	.1356	.1466
3	$2\frac{1}{3}$	7.3304	.2143	.1364	.2728	.1578	.2942	.1328	.1436
2 5	$2\frac{1}{2}$	7.8540	.2000	.1273	.2546	.1473	.2746	.1240	.1340
3 8	$2\frac{2}{3}$	8.3776	.1875	.1194	.2387	.1381	.2575	.1163	.1256
11	$2\frac{3}{4}$	8.6394	.1818	.1158	.2313	.1340	.2498	.1127	.1218
1 3	3	9.4248	.1666	.1061	.2122	.1228	.2289	.1033	.1117
$\frac{5}{16}$	$3\frac{1}{5}$	10.0531	.1562	.0995	.1989	.1151	.2146	.0969	.1047
3 10	$3\frac{1}{3}$	10.4719	.1500	.0955	.1910	.1105	.2060	.0930	.1005
2/7	$3\frac{1}{2}$	10.9956	.1429	.0909	.1819	.1052	.1962	.0886	.0957
1/4	4	12.5664	.1250	.0796	.1591	.0921	.1716	.0775	.0838
9	$4\frac{1}{2}$	14.1372	.1111	.0707	.1415	.0818	.1526	.0689	.0744
1 5	5	15.7080	.1000	.0637	.1273	.0737	.1373	.0620	.0670
3 16	$5\frac{1}{3}$	16.7552	.0937	.0597	.1194	.0690	.1287	.0581	.0628
2 11	$5\frac{1}{2}$	17.2788	.0909	.0579	.1158	.0670	.1249	.0564	.0609
1 6	6	18.8496	.0833	.0531	.1061	.0614	.1144	.0517	.0558
2 13	$6\frac{1}{2}$	20.4203	.0769	.0489	.0978	.0566	.1055	.0477	.0515
1 7	7	21.9911	.0714	.0455	.0910	.0526	.0981	.0443	.0479
15	$7\frac{1}{2}$	23.5619	.0666	.0425	.0850	.0492	.0917	.0414	.0446
1 8	8	25.1327	.0625	.0398	.0796	.0460	.0858	.0388	.0419
1 9	9	28.2743	.0555	.0354	.0707	.0409	.0763	.0344	.0372
10	10	31.4159	.0500	.0318	.0637	.0368	.0687	.0310	.0335
16	16	50.2655	.0312	.0199	.0398	.0230	.0429	.0194	.0209
1 20	20	62.8318	.0250	.0159	.0318	.0184	.0343	.0155	.0167

GEAR WHEELS.

TABLE OF TOOTH PARTS—DIAMETRAL PITCH IN FIRST COLUMN.

Diametral Pitch.	Circular Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.
P	P'	t	8	D"	s+f.	D''+f.
$\frac{1}{2}$	6.2832	3.1416	2.0000	4.0000	2.3142	4.3142
<u>3</u> 4	4.1888	2,0944	1.3333	2.6666	1.5428	2.8761
1	3.1416	1.5708	1.0000	2.0000	1.1571	2.1571
$1\frac{1}{4}$	2.5133	1.2566	.8000	1.6000	.9257	1.7257
$1\frac{1}{2}$	2.0944	1.0472	.6666	1.3333	.7714	1.4381
$1\frac{3}{4}$	1.7952	.8976	. 5714	1.1429	.6612	1.2326
$\frac{1}{2}$	1.5708	.7854	.5000	1.0000	.5785	1.0785
$2\frac{1}{4}$	1.3963	.6981	.4444	.8888	.5143	.9587
$2\frac{1}{2}$	1.2566	.6283	.4000	.8000	.4628	.8628
$2\frac{3}{4}$	1.1424	.5712	.3636	.7273	.4208	.7844
3	1.0472	.5236	.3333	.6666	.3857	.7190
$3\frac{1}{2}$	.8976	.4488	.2857	. 5714	.3306	.6163
4	.7854	.3927	.2500	.5000	.2893	. 5393
5	.6283	.3142	.2000	.4000	.2314	.4314
6	.5236	.2618	.1666	.3333	.1928	.3595
7	.4488	.2244	.1429	.2857	.1653	.3081
8	.3927	.1963	.1250	.2500	.1446	.2696
9	.3491	.1745	.1111	.2222	.1286	.2397
10	.3142	.1571	.1000	. 2000	.1157	.2157
11	.2856	.1428	.0909	.1818	.1052	.1961
12	.2618	.1309	. 0833	.1666	.0964	.1798
13	.2417	.1208	.0769	.1538	.0890	.1659
14	.2244	.1122	.0714	.1429	.0826	.1541

### TABLE OF TOOTH PARTS—Continued.

DIAMETRAL PITCH IN FIRST COLUMN.

Diametral Pitch.	Circular Pitch.	Thickness of Footh on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.
P.	P'.	t.	8.	D''.	8+f.	D''+f.
15	.2094	.1047	.0666	.1333	.0771	.1438
16	.1963	.0982	.0625	.1250	.0723	.1348
17	.1848	.0924	.0598	.1176	.0681	.1269
18	.1745	.0873	. 0555	.1111	.0643	.1198
19	.1653	.0827	.0526	.1053	.0609	.1135
20	.1571	.0785	.0500	.1000	.0579	.1079
22	.1428	.0714	.0455	.0909	.0526	.0980
24	.1309	.0654	.0417	.0833	.0482	.0898
26	.1208	.0604	.0385	.0769	. 0445	.0829
28	.1122	.0561	.0357	.0714	.0413	.0770
30	.1047	.0524	. 0333	.0666	.0386	.0719
32	.0982	.0491	.0312	.0625	.0362	.0674
34	.0924	.0462	.0294	.0588	.0340	.0634
36	.0873	.0436	.0278	.0555	.0321	.0599
38	.0827	.0413	.0263	.0526	.0304	.0568
40	.0785	.0393	.0250	.0500	.0289	.0539
42	.0748	.0374	.0238	.0476	.0275	.0514
44	.0714	.0357	.0227	.0455	.0263	.0490
46	.0683	.0341	.0217	.0435	.0252	.0469
48	.0654	.0327	.0208	.0417	.0241	.0449
50	.0628	.0314	.0200	.0400	.0231	.0431
56	.0561	.0280	.0178	.0357	.0207	.0385
60	.0524	.0262	.0166	.0333	,0193	.0360

Comparative Sizes of Gear Teeth. Involute.

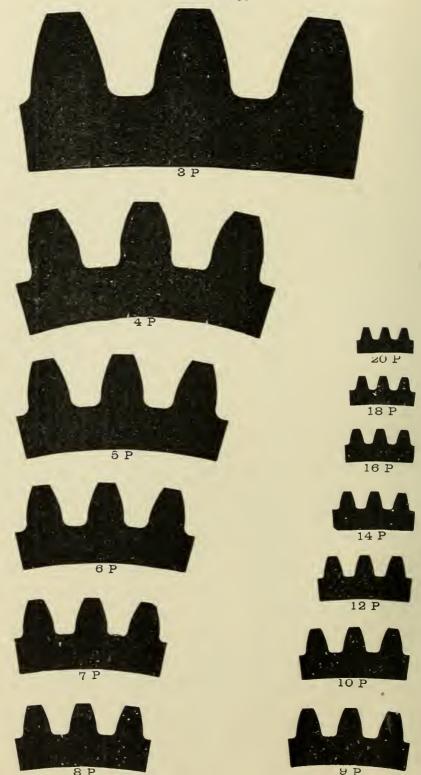
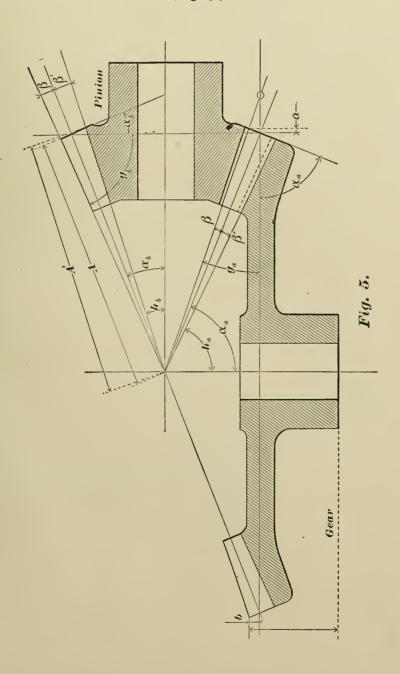


Fig. 4.

## CHAPTER III.

## BEVEL GEARS.—AXES AT RIGHT ANGLES.

(Fig. 5.)



#### FORMULAS.

 $N_a = \begin{cases}
N_a = \begin{cases}
N_b = \end{cases}$  Number of teeth  $\begin{cases}
\text{gear.} \\
\text{pinion}
\end{cases}$ 

P = diametral pitch.

P' = circular pitch.

 $\alpha_a = \alpha_b$  center angle = angle of edge  $\beta$  gear.  $\alpha_b = \beta$  or pitch angle  $\beta$  pinion.

 $\beta$  = angle of top.

 $\beta' = \text{angle of bottom}.$ 

 $\begin{cases}
 g_a = \\
 g_b = 
 \end{cases}$  angle of face  $\begin{cases}
 \text{gear. pinion.} \\
 h_a = \\
 h_b = 
 \end{cases}$  cutting angle  $\begin{cases}
 \text{gear. pinion.} \\
 \text{pinion.} 
\end{cases}$ 

A = apex distance from pitch circle.

A' = apex distance from large bottom of tooth.

d = pitch diameter.

d' = outside diameter.

s = addendum and module.

t =thickness of tooth at pitch line.

f = clearance at bottom of tooth.

D'' =working depth of tooth.

D'' + f = whole depth of tooth.

a = diameter increment.

b =distance from top of tooth to plane of pitch circle.

F = width of face.

$$\tan \alpha_a = \frac{N_a}{N_b} - ; \quad \tan \alpha_b = \frac{N_b}{N_a} ;$$

$$\tan \beta = \frac{2 \sin \alpha}{N} ; \text{ or } \quad \tan \beta = \frac{s}{A} .$$

$$\tan \beta' = \frac{\sin \alpha}{N} \left(2 + \frac{\pi}{10}\right) = \frac{2 \cdot 314 \sin \alpha}{N} ; \quad \tan \beta' = \frac{s + f}{A} ;$$

$$g_a = 90^\circ - (\alpha_a + \beta) ; g_b = 90^\circ - (\alpha_b + \beta)$$

$$h = \alpha - \beta' \qquad (See Note, page 6g.)$$

$$A = \sqrt{\frac{N_a}{2P}} + \frac{N_b}{2P}^2$$

$$A = \frac{N}{2 P \sin \alpha}$$

$$A' = \frac{A}{\cos \beta'} \qquad A' = \frac{N}{2 P \sin \alpha \cos \beta'}$$

$$A = \frac{\frac{1}{2} d'}{\sin (\alpha + \beta)} \cos \beta$$

$$P = \frac{N}{2 A \sin \alpha}$$

$$d = \frac{N}{P} \text{ or } = \frac{N P'}{\pi} \qquad d' = d + 2 \text{ a}$$

$$2 \text{ a } = 2 \text{ s } \cos \alpha \qquad (See page 2o.)$$

$$b = a \tan \alpha \quad \begin{cases} a \text{ for gear} = b \text{ for pinion} \\ a \text{ for pinion} = b \text{ for gear} \end{cases}$$

$$P = \frac{\pi}{P'} \qquad P' = \frac{\pi}{P}$$

$$s = \frac{1}{P} = \frac{P'}{\pi} = .3183 P' \quad s = A \tan \beta$$

$$s + f = .3685 P' \qquad s + f = A \tan \beta'$$

$$s + f = \frac{1}{P} \left(1 + \frac{\pi}{20}\right) \qquad D'' = 2 \text{ s}$$

$$t = \frac{P'}{2} = \frac{\pi}{2P} \qquad f = \frac{1}{10}t$$

$$F = \frac{4}{P} + \frac{A}{7} \text{ or } = 2 \text{ P' to } 3 \text{ P'}$$

Note.—Formulas containing notations without the designating letters a and b apply equally to either gear or pinion. If wanted for one or the other, the respective letters are simply attached.

## BEVEL GEARS WITH AXES AT ANY ANGLE.

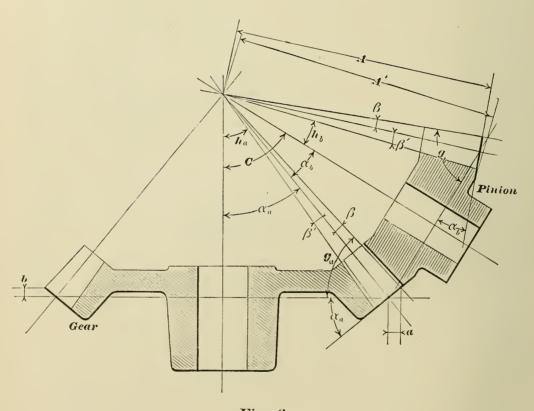


Fig. 6.

#### FORMULAS.

C =angle formed by axes of gears.

P = diametral pitch.

P' = circular pitch.

 $\alpha_a = \begin{cases}
\alpha_a = \\
\alpha_b =
\end{cases}$  angle of edge = pitch angle  $\begin{cases}
\text{gear.} \\
\text{pinion.}
\end{cases}$ 

 $\beta$  = angle of top.

 $\beta'$  = angle of bottom.

 $\begin{cases}
 g_a = \\
 g_b =
 \end{cases}$  angle of face  $\begin{cases}
 \text{gear.} \\
 \text{pinion.}
\end{cases}$ 

 $\begin{array}{c} h_a = \\ h_b = \end{array}$  cutting angle  $\begin{cases} \text{gear.} \\ \text{pinion.} \end{cases}$ 

A = apex distance from pitch circle.

A' = apex distance from large bottom of tooth.

d = pitch diameter.

d' = outside diameter.

a = diameter increment.

b =distance from top of tooth to plane of pitch circle.

NOTE.—The formulas for tooth parts as given on page 5 apply equally to these cases.

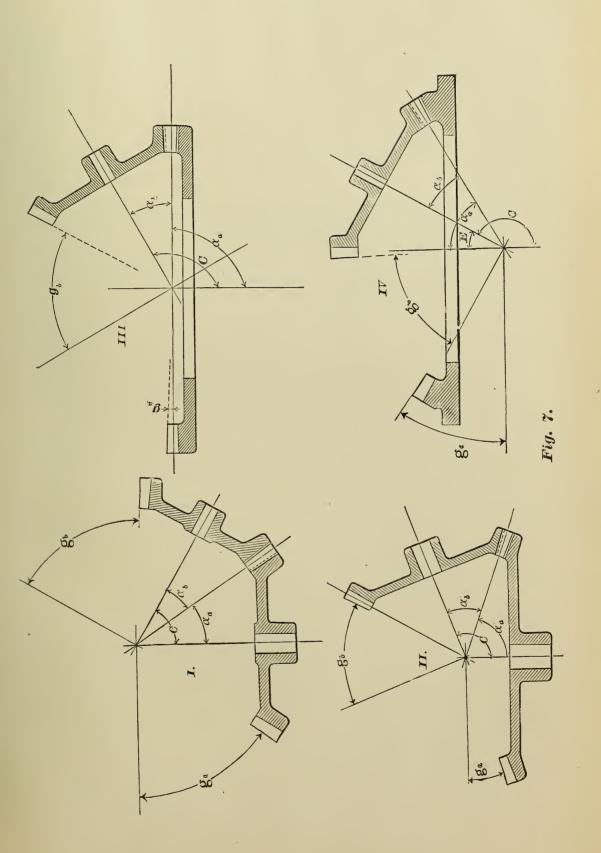
$$\tan \alpha_a = \frac{\sin C}{\frac{N_b}{N_a} + \cos C}; \text{ or } \cot \alpha_a = \frac{N_b}{N_a \sin C} + \cot C$$

$$\tan \alpha_b = \frac{\sin C}{\frac{N_a}{N_b} + \cos C}; \text{ or } \cot \alpha_b = \frac{N_a}{N_b \sin C} + \cot C$$

Note.—The above formulas are correct only for values of C less than 90°. If C is greater than 90°, consult page 18.

tan 
$$\beta = \frac{2 \sin \alpha}{N}$$
; or tan  $\beta = \frac{s}{A}$ ;  
tan  $\beta' = \frac{\sin \alpha \left(2 + \frac{\pi}{10}\right)}{N} = \frac{2.314 \sin \alpha}{N}$ ; tan  $\beta' = \frac{s+f}{A}$ ;  
 $g_{\alpha} = 90^{\circ} - (\alpha_{\alpha} + \beta)$  for Cases I and II.  
 $g_{\alpha} = \beta$ , for Case III.  
 $g_{\alpha} = 90^{\circ} - (\alpha_{\alpha} - \beta)$  for Case IV.  
 $g_{b} = 90^{\circ} - (\alpha_{b} + \beta)$   
 $h = \alpha - \beta'$  (See page 69.)  
 $A = \frac{N}{2 \text{ P sin } \alpha}$   
 $A' = \frac{A}{\cos \beta'}$   
 $d = \frac{N}{p} \text{ or } = \frac{N \text{ P'}}{\pi}$   
 $d' = d + 2 \text{ a}$  (for Cases I and II.  
 $d' = d$ , for gear in Case III.  
 $d' = d - 2 \text{ a}$ , for gear in Case IV.  
 $2 \text{ a} = s \cos \alpha$   
 $b = s \sin \alpha$ 

Note.—Formulas containing notations without the designating letters a and b apply equally to either gear or pinion. If wanted for one or the other, the respective letters are simply attached.



The formulas given for  $\alpha_a$  and  $\alpha_b$  (when C.  $N_a$  and  $N_b$  are known) undergo some modifications for values of C greater than 90°.

For bevel gears at any angle but 90° we may distinguish four cases; C,  $N_a$ ,  $N_b$  being given.

I. Case. See pages 14 and 16.

II. Case. C is greater than 90°.

$$\tan \alpha_a = \frac{\sin (180 - C)}{\frac{N_b}{N_a} - \cos (180 - C)}; \quad \tan \alpha_b = \frac{\sin (180 - C)}{\frac{N_a}{N_b} - \cos (180 - C)}$$

III. Case.  $\alpha_a = 90^\circ$ ;  $\alpha_b = C - 90^\circ$ 

IV. Case.

$$\tan \alpha_a = \frac{\sin E}{\cos E - \frac{N_b}{N_a}}; \quad \tan \alpha_b = \frac{\sin E}{\frac{N_a}{N_b} - \cos E}$$

For an example to apply to Case III., the following condition must be fulfilled:

$$N_a \sin (C - 90^\circ) = N_b$$

To distinguish whether a given example belongs to Case II. or case IV., we are guided by the following condition:

Is:  $N_a \sin (C - 90^\circ) \begin{cases} smaller \text{ than } N_b, \text{ we have Case II.} \\ larger \text{ than } N_b, \text{ we have case IV.} \end{cases}$ 

### UNDERCUT IN BEVEL GEARS.

By undercut in gears is understood a special formation of the tooth, which may be explained by saying that the elements of the tooth below the pitch line are nearer the center line of the tooth than those on the pitch line. Such a tooth outline is to be found only in gears with few teeth. In a pair of bevel gears where the pinion is low-numbered and the ratio high, we are apt to have undercut. For a pair of running gears this condition presents no objection. Should, however, these gears be intended as patterns to cast from, they would be found useless, from the fact that they would not draw out of the sand. We have stated on page 2 (see Fig. 1) that the base of our involute system is the 14½° pressure angle. If a pair of bevel gears with teeth constructed on this basis have undercut, we can nearly eliminate the undercut—and for the practical working this is quite sufficient-by taking as a basis for the construction of the tooth outline a pressure angle of 20°.

The question now is: When do we, and when do we not have undercut? Let there be:

N = number of teeth in gear. n = number of teeth in pinion.

$$\frac{n\sqrt{N^2+n^2}}{N} = p$$

where we have undercut for p less than 30.

This formula is strictly correct for epicycloidal gears only. It is, however, used as a safe and efficient approximation for the involute system.

### DIAMETER INCREMENT.

2 a.

Rule.—The ratio being given or determined, to find the outside diameter divide figures given in table for large and small gear by pitch (P) and add quotient to pitch diameter.

RAT	10.	GEARS.		RATIO.		GE.	ARS.	RATIO.		GEARS.		
Large		Large	Small			Large	Small				Small	
1.00 1.05	1:1	1.41 1.37	1.41 1.42	1.65 1.67	5:3	1.05 1.03	1.70 1.72	4.40 4.50	9:2	.45	1. <sup>9</sup> 4 1.95	
$ \begin{array}{c} 1.07 \\ 1.10 \\ 1.11 \end{array} $	10:9	1.36 1.35 1.34	1.43 $1.44$ $1.46$	$     \begin{array}{c c}       1.70 \\       1.75 \\       1.80 \\    \end{array} $	7:4 9:5	1.01 .99 .97	1.73 $1.74$ $1.75$	$egin{array}{c} 4.60 \\ 4.80 \\ 5.00 \\ \hline \end{array}$	5:1	.42 .41 .39	$   \begin{array}{c}     1.96 \\     1.96 \\     \hline   \end{array} $	
1.12 1.13 1.14	9:8 8:7	1.33 1.33 1.32	$   \begin{array}{c}     1.46 \\     1.47 \\     1.49 \\     \hline   \end{array} $	1.85 1.90 1.95	0.1	.95 .93 .91	1.76 1.77 1.78	5.20 5.40 5.60		.38 .37 .36	1.96 $1.96$ $1.97$	
1.15 1.16 1.17 1.18	7:6	1.31 1.30 1.30	1.50 $1.51$ $1.52$	$egin{array}{c} 2.00 \ 2.10 \ 2.20 \ 2.25 \ \end{array}$	2:1	.89 .87 .84	1.79 1.80 1.81	5.80 6.00 6.20	6:1	.34	1.97 1.97 1.97	
1.19 $1.20$ $1.23$	6:5	1.29 1.28 1.28 1.27	1.53 $1.53$ $1.54$ $1.55$	$\begin{bmatrix} 2.25 \\ 2.30 \\ 2.33 \\ 2.40 \end{bmatrix}$	9:4	.82 .80 .78 .76	1.82 1.83 1.84 1.85	$\begin{bmatrix} 6.40 \\ 6.60 \\ 6.80 \\ 7.00 \end{bmatrix}$	7:1	.31 .30 .29 .28	1.97 1.97 1.98 1.98	
1.25 $1.27$ $1.29$	5:4 9:7	1.25 1.25 1.24	1.56 1.57 1.58	$\begin{bmatrix} 2.40 \\ 2.50 \\ 2.60 \\ 2.67 \end{bmatrix}$	5:2 8:3	.75 .73 .71	1.86 1.86 1.87	$\begin{bmatrix} 7.20 \\ 7.40 \\ 7.60 \end{bmatrix}$	7:1	.27 .27 .26	1.98 1.98 1.98	
1.30 1.33 1.35	4:3	1.22 1.20 1.18	1.59 $1.60$ $1.61$	$\begin{bmatrix} 2.70 \\ 2.80 \\ 2.90 \end{bmatrix}$	0.0	.69 .67 .65	1.87 1.88 1.89	7 80 8 00 8 20	8:1	$   \begin{array}{c}     .26 \\     .25 \\     .24   \end{array} $	1.98 1.98 1.98	
1 37 1.40 1.43	7:5 10:7	1.17 1.16 1.15	1.61 1.62 1.63	3.00 3.20 3.33	3:1	.63 .60 .58	1.91 1.92 1.92	8.40 8.60 8.80		.24	1.98 1.98 1.98	
1.45 $1.50$ $1.53$	3:2	1.13 1.11 1.10	1.65 1.66 1.67	3.40 3.50 3.60	7:2	.56 .54 .52	1 92 1 93 1 93	$ \begin{vmatrix} 9.00 \\ 9.20 \\ 9.40 \end{vmatrix} $	9:1	.22	1.99 1.99 1.99	
1 55 1.58 1.60	8:5	1.09 $1.08$ $1.07$	1.67 1.68 1.68	3.80 4.00 4.20	4:1	.50 .49 .47	1.94 1.94 1.94	9.60 $9.80$ $10.00$	10:1	.21	2.00 2.00 2.00	
	0.0	2.01	1.00	1.20			1.01	10.00	10.1	. 20	~ 00	

NOTE.—To be used only for bevel gears with axes at right angle.

# TABLES FOR ANGLES OF EDGE AND ANGLES OF FACE.

The following four tables have been computed for the convenience in calculating datas for bevel gears with axes at right angle. They *do not hold* good for bevel gears with axes at any other angle.

To use the tables the number of teeth in gear and pinion must be known.

Having located the number of teeth in the gear on the horizontal line of figures at the top of the table, and the number of teeth in the pinion on the vertical line of figures on the left-hand side, we follow the two columns to the square formed by their intersections.

The two angles found in the same square are the respective angles for gear and pinion. The tables are so arranged that the angle belonging to the gear is always placed above the angle for the pinion.

### TABLE 1

ANGLE OF EDGE.
GEAR.

		GEAR.													
	41			38			35	34			31			28	27
12	73°41			72 28							68°50				
_		71 59		17°32 71°7											23°s
13	17°35			18 53											
14	71 9	70 43	70 15	69 46	69 16	68 45	68,15	67°37	67°0	66 ss	65 42	64 59	64°14	63°26	62's
14				20°4											
15	69°54	20 34		68 28											
16	68°41	68°12′	67 42	67°10	66°37	66° 2′	65°26	64 48	64°8′	63 %	<b>62°</b> 42	61°56	61°7	60°15	59 8
10	_			22 50			24°34	25°12							
17	67 29 22 31		66°27	65°54 24°6					62°45					58 44	
10	66°18	6546		64 39											
18				25°21											
19	65°8		64°2				61°30								
13				26°84											
20	64° 0′			62°14 27°46											
21				61°4′											
21	27°7'	27°42	28°19	28°55	29°35	30°15	30°58	31 42	32°28	33°17	34°7′	35° o′	35 55	36°53′	37°5
22	61°47			59 56											
22	28 13			30°4′ 58°49											
23	60°42			31 11											
24	5939			57°44											
<u> </u>				3216											
25				56 40										48°14	
	A 70 .			33°20 5537								49°5			
26				34 23											
27	56°38	55°s	55°18	54°36	53°53	53°7	52 21	5133	50°43	49°51	48 57	48°0	47°3	46°2	15
21	33 22	34 1	34 42	35°24	36 7	36 53	37 39	38 21	39 17	40 9	41 3	42 0	42 57	43 58	-
28	5540	55°0	54 19	53°37 36°23	22.23	52°8	3900	50 32	49 41	48 48	47 55	46 58	46 0	45°	
20				52-39									4	<del>                                     </del>	j
29				37°21										}	
30				51 42											
				38°18									j		
31				39 12											
_	52°2	51 20	50°38	49°54	49°9	48 22	47°34	46 44	45°53	4.50	1	,			
32				40°6						45	J				
33	51 10	5029	49 46	49°2	48 16	47 29	4641	4551	45°						
24	50 20	4938	48°55	48°11	47 25	42 21 46°38	45 50	44 9		j					
34	0340	40 22	41 2	141 43	P4 2 35	43 22	44 10	45	]						
25				47°21											
00		480		42 39		1		<u>}</u>							
36				43 27											
27	47°56	47°14	46°30	4546	A F°		,								
3/	42°4			44 4	73	j									
38		43 32													
00	42 50			-	J										
39	43 34	44 17													
AC	4542	100	-												
40	44 18	1,3	J												
	45°														

# TABLE 1.—(Continued.)

ANGLE OF EDGE.

GEAR.

	26	25	24	<b>2</b> 3	22	21	20	19	18	17	16	15	14	13	12	
12					61°23 28°37										45°	
13	63,56	62°31	61 33	60°31	59 25	58 4	56°58	55°37	54°10	52°36	50°54	49°5	47°7	45°		i
14		_		_	30°35 57°32								42°53		}	
14					32°28′ 55°43′								45			
12	29 59	30 58	32°0′	33"7"	34 17	35 32	36°53	38°18	39°48	41 25	43°9	45°				
16	31°37		3341	34°49	53°58 36°2	37°18′	38 40	40°6	41°38	43°16						
17		55°47 34°13			52° 18' 37°42'				46°36 43°22							
18		54° 15'			50 43 39 17				45°							
19	53°sı′		51°38	50°26	49°11′ 40°49′	47°52	46°28	Δ5°		,						
20	52 26	51°20	50°12'	48 59	47°43	46°24	450	-	`							
21	51°4'	49°58		47°36	42°17 46°20	4E°	-									
22		40°2′ 48°39	41 kg		43°40		j									
22	40°14		42°31	43°44	45°											
23	41 30	42 37	43°47	45°												
24	42°43	46°10 43°50														
25	46°7 43°53	45°														
26	45°															

$$an \ lpha_a = rac{ extbf{N}_a}{ extbf{N}_b}$$
  $an \ lpha_b = rac{ extbf{N}_b}{ extbf{N}_a}$ 

(See page 13.)

## TABLE 2.

# Angle of Edge. Gear.

Ī		72	71	70	69	68	67	66	65	64	63	62	61	60	59	58	<b>57</b>
	12	80°33 9°27	80°26 9°35	80°16 9°44	80°8 9°52	79°59	79 st	79 42	79°32 10°26			79° 3′ 10° 57		78°41 11°19	78°36	78°19 11°41	78°7′ 11°53
I	13	79°46	79°37	79°29 10°31	79°20	79°11 10°49	79° 1	78°51	78°41 11°19	78°3í	78°20	_	77°58	77°46		77°22	77°9′ 12°51
ŀ	4	79°0	78°51	78°41	78°32 11°28	78°22	78°11		77°51	77°46 12°20	77 28	77°17	77°5	76°52 13°8	76°39		76 12
ŀ	15	78°14	78°4	77°54	77°44	77°34	77°23	77°12	77°0′	76°48	76°36	76°24	76°11	75 58	75°44	75°36	75°15
1	16	77°28	11°56	12°6	12° 16 76° 57	12°26 76°45	76°34	76°22		75°58		75°32	75°18	14° 2'	74°49	14 30 74°35	14 45 74°19
-	17	12°32 76°43	12°42	12°53	13° 3′ 76° 10	13°15 75°58		13°38 75°33	13°50	14° 2'	14° 15			14 56 74° 11	15°11 73°56	15°25 73°40	15°41 73°24
-	1/	13°17	13°28 75°46	13°39	13°50 75°21	14° 2'		14° 27 74° 45	14°39	14°52		15°20 73°49	15°35 73°33	15°49 73°18		16°20 72°45	16°36
-	18	14° 2′ 75° 13	14°14	14°25	14° 37	14°50			15°29	_	15°52		16°27			17°15	17°31
-	19	14°47	14°59	15°11	15°24	15°37	15°50	16°4	16°18	16 32	16°47	17° 2	17°เธ	17°34	17°51	18° 8′	18°26
	<u> 20</u>	74°29 15°31	74 16 15 44	74°3′ 15°57	73°50 16°10	73°37 16°23			72°54	72°59	17 37	17 53	_	71°34 18°26	18°44	_	70°40 19°20
	21	73°45	73°92 16°28	73°18	73°4 16°56	72°50 17°10	72°36 17°24		72°6		18 26	18°43	71° oʻ 19° oʻ	70°43 19°17	70°24 19°36		69°46 20°24
	22	73° i′ 16°59	72°47	72°33	72°19 17°41	72°4′ 17°56	71°49 18°11	71°34 18°26	71° 18 18°42	71° 2 18° 58	70°45			69°52 20° 8	69°33 20°27	المناالساسل	68°54 21°6
	23	72°ri 17°43	72°3 17°57	71°49 18°11	71°34 18°26	71° m 18°41	71°3	70°47	70°90 19°20	70°4	69°57 20°3	69°39 20°21		69°2	68°42 21°18	68 22 21 38	68° 2'
H	24	71°34 18°26	71°19 18°41	71°5′ 18°55′	70°49 19°11	70°34 19°26	70017	70° 1′	69°44 20°16	69°26	<b>69°</b> 9′	68°50	68°31	68°12 21°48	67°52	67°31 22°29	67°10 22°50
	25	70°51	70°36	70°21	70° 5	69°49	69°32	69°15	68°57 21°3	68°40	68°21	68° 3'	67°43	67 23	67°2′	66°41	66°19
	26	19°9	19 24 69 53		69°21	20 II 69°4	68°46	20°45 68°30	68°12	67°54	67°34	67°15	22°17		66°13	65 51	23 41 65 29
	27	19°51	20 7 69°10		20°39 68°38		68°3	21°30	67°26	67°8		66°28		65°46	23 47 65°25		24°31 64°39
	20	20°33 68°46	20°50		21°22 67°55				22°34 66°42	22 52 66°22			23 53 65°21	64°59		24°58	25°21
	28	21°15	21°31		22° 5′ 67° 12	22°23	_				23°58 65°16		24°39		25°23 63°50	25°46 63°26	26°10
	29			22°30	22°48 66°30	23 6	23 24	23°43	24° 3′	24 23	24 44	25° 5	25°26	25°48		26°34	السالانطاط
	30	22°37	22°54	23°12	23 30	23 48	24° 8	24°27	24 46	25°7′	25°28	25°50	26° 11	26 34	26°57	27°21	27 46
	31	66°42 23°18	66 25 23 35	_	24 12	24° 31			25°30	2551		26°34	63°3 26°57		27 42	_	61°28 28°32
	32	66°2′ 23°58	65 44 24 16		65° 7′ 24° 53	64 48 25°12	64°28 25°32		63°47 26°13	26 34	26°56	62°42 27°18	62°19 27°41	61°56 28°4	61°32 28°28	61°7 28°53	60°41 29° 19
	33	65°23 24°37			64°26 25°34		63°47 26°13	63°26 26°34		62°43 27°17		61°58 28°2	61°35 28°25	61°11 28°49	60°47 29°13	60°21 29°39	59°56 30°4
	34				63°46 26°14							61°15 28°45					59° 11 30° 49
Ì	35	64° 5'	63°45	63°26	63° 6	62°46	62°25	62°4'	61°42	61°19	60°57	60°33	60°9	59°45	59°19	58°53	58°27 31°33
ı	_		0 '	0 '	62°27 27°33	200	-	e.0 '	C.0 1	0	E 00 1	Eo°.	E-0-1	50°-	ro° '	- 0°	
1	<del>37</del>	62 48	162 28	1628	[ <b>6</b> ] 48	61 27	615	60 44	160 211	22 25	159 351	59 101	2846	28 SO	3/ 5A	5/ 28	2/11
ŀ	00	62°11	61°51	61°30	28°12	60049	60°26	60°4	59°41	59°19	58°54	58°30	58 5	57°39	57°13	56 46	56°19
ŀ		61°33	61°15	60°53	28 51 60°31	60°10	59°48	59°25	59 2	58 39	58°14	57°50	57°24	56 58	56°32	56°6	55°37
ŀ	<u>39</u>	28° 27	2847	29°7	29°29 59°53	29°50 59°32	30°12	30°35	30°58 58°24	31°21	31°46 57°35	32°10	32°36	33° 2'	33 æ 55 sz	33 54 55°24	34°23 54°57
-	40	29°3′	294	2945	30°7′ 59°17	30°26	30°50	31°13	31°36	32°0′	32°25	32°50	33 16	33 41	34° 8	34 35	35°3′
	41	29°40	30°0	30°21	30°43	31°5	31°28	31°51	32 15	32°39	33°3	33 26	33° 54	34°21	34 48	35°16	35 44
	42	30 15	3036	3057	31°20	31 42	32 5	32 28	35.25	33 17	33 41	34 7	34 33	350	35 27	35 55	36 23

アンファ

# TABLE 2.—(Continued.)

ANGLE OF EDGE. GEAR.

								GEA	K.							
ſ		56	55	54		<b>52</b>						46			43	42
	12	12°6	77 42	12 32	12 45	13°0′	13 14	1300	13 46	14°2'	14019	75°23 14°37	14 56	15 15	74°25 15°35	15°57
Ī	13	76°56	76°42 13°18	76°28	76°13 13°47	75°56 14° 2'	75°46 14°18	75°26	75°8′ 14°52	74°si 15°9	74°32 15°28	74°13	73°53	73°32 16°28	73°11 16°49	72°46 17°12
İ	14	75°58 14° 2′	75°43	75°28	75°12	74°56	74°39	74°21	74°3′	73°44	73°25	73°4	72 43	72°21	71°58 18°2	71°34
ŀ	15	75°0′	74°44	74°29	74° 12	73°55	73°37	73°18	72°59	72°39	72°18	71°56	71°34	71°16	70°46	7021
ł	16	15° o'	7341	73 36	73°12	72°54	72°35	7205	71°55	71°34	71° 12	70°49	70 26	70° 1	19°14 69°35	69°9′
+	_	73°7	72°49	72°31	72°13	71°54	71°34	71°13	70°52	70°30	70°7	69 43	69°17	<b>68</b> ,25	20°25	67°58
-	<u>17</u>														21°34 67°17	
-	18	17 49	18°7	18 26	18 45	19° 6	19°27	19°48	20°10	20°34	20°57	21°25	21 48	22 15	22.43 66°10	23 12
	19	18 45	19° 1	19°23	19 43	20°4'	20°26	20 48	21°12	21°35	22 1	22°26	22 54	23 22	23 50	24 21
	<u> 20</u>	19°39		20° 19	2041	21°3′	21°25	21 48	22 12	22°37	23° 3′	23°30	23°58	24 27	65° 3 24° 57	25 28
	21														63°58 26°2	
	22														62°54 27°6	
Ī	23	67°41	67°18	66°55	66°32	66° 8	65°44	65°18	64°51	64°24	63°55	63°26	62°56	62°24	61°sz 28°8	6118
Ì	24	66°48	66°26	66° 2′	65°38	65°14	64°48	64°22	63°54	63°26	62 57	62°27	61°56	61°23	60°50 29°10	60°15
ŀ	25	65°57	65°33	65° 9′	64°45	64° 20	63°53	63°26	62°58	62°29	61°59	61°29	60°57	60°24	59°50	59°14
۶ŀ	25 26	65°6	6A 42	64°18	63,25	63°26	62°59	62°31	62°3	61°33	61°3	60°31	5985	59°25	30°10 58°50	58°H
, l															31° 10 57°53	
	27	25 44	26 9	26°34	27°0'	27°26	27 54	28 22	28°52	29 22	29 53	3025	30 58	31 32	32° 7	32 44
ŀ	28	26°34	26 59	27°24	27°51	28 18	28°46	29°15	29 45	30°15	30 47	31°20	31°53	32°28	33°4	33 41
-	29	27 23	27°48	28°15	2841	29 9	29 37	30°7	30 37	31° 8	31°41	32° 14	32 48	33 23	34° 0	34 87
	<u>30</u>	28°11	28 37	29°3′	2931	29 59	30°28	30 58	31°28	32°0′	32 23	33° 7	33°41	34° 17	55°s' 34°53	35 32
	31	2858	29 24	29°54	30 19	30°46	31 16	31 48	32°19	32°52	33°24	33 59	34 34	35 10	54° 12 35°48	36°26
	32	60 is	59 48 30 12	59°21	58°52 31° 8′	58°34 31°26	57°54	57°23 32°37	56°52 33°8	56°19	55°45	55°11′ 34°49′	54°35	53°58	53°21 36°39	52°42 37°18
-	33	59°29	59°2′	58 34	58°5	57°36	57°6	56°34	56°2'	55°30	54° 56	54°21	5325	53°8	52°29 37°3í	51°56
ı	34	58 44	58 16	57°48	57°19	56°49	56 19	5547	55°15	54°41	54° 7	53°32	52°52	52°18	51°40 38°20	51° 0′
ŀ	25 25	58° o	57°32	57°3′	56°33	56°3′	55 32	55°0	54°28	53°54	53°20	52°44	52°8	51°30	50°51	50°12
ŀ	36	57 16	5648	5619	5549	55°18	54°47	54°15	53°42	53° 6	52°33	51°57	51°20	50 43	39° 9′ 50°4′	49°24
-	_														39°56	
1	37														40°43 48°32 41°29	
_	_	34°9	34 39	35° 8	35°37	36°9	36 42 52°0	37°14	3748	38 22	38 57	39°33	40°11	40°49	41° 29 47° 46	42 8
ŀ	39	34 51	35°21	35 50	3621	36 53	37 24	37 57	38 31	39 6	39 41	40°18	40 55	41°33	42 12	42 53
4	40	35°32	36 2	36 32	37° 2	37°34	38 6	38 40	39 14	39°48	40 24	41°1	41°38	42°16	47°5 42°55	43°36
-	41	3612	3643	37°12	37°44	38 15	38 48	39 21	39 55	40°36	41° 6	41 43	42°20	42°59	46°22 43°38	44 19
	10	53°8'	52 38	52°8′	51°36	51°4	50 32	49°58	49 24	48 49	48°13	47°36	46°59	46°20	45 40	
~	_										-					

TABLE 3.

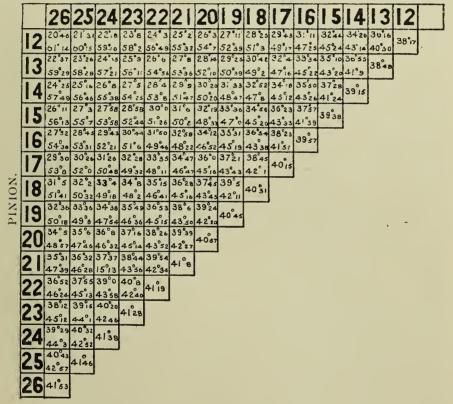
# ANGLE OF FACE. GEAR.

							GE	AR.		,	,—				,
	41	40	39	38	37	36	35	34	33	32	31	30	29	28	27
12							15 49	15 15	1643	17'18	1743	18° 15	18°51	18 27	20 5
	70°59	15°.17	15°39	16% 1	16°25	16°51	67°59	1746	18 16	1848	19 21	19°57	6333 20°02	21011	62° 9
13	69°45	69°15	68°47	68°15	67°43	67°9	66°33	65°56	65°16	64°34	63°51	63°5	6214	61°23	6021
4							18°45 65°9								
15	17020	17°53	18°18	18 44	19 11	19°40	20°11	2044	81,13	21°53	2281	23°10	23'51	24°35	25°20
13	18 42	66 45	66°14	6540 20°3	65°3	64 26	6349	63°8	6224	61°39	60°61	2448	59°a	58 13	5714
16	66°4	65°33	64°51	64°28	63°46	63°7	628	6145	61°01	60°14	59°25	58°34	574º	56°42	55%
17							22°87 61°9								
18	210	2137	22°6	2298	23°s	23°43	24°18	24°66	2534	2615	26°57	2742	28 29	29°18	30°9
							59°52 25°37								
19	6236	62°1	61024	6044	60°4	5921	58°37	57°51	57°4	56°14	5582	54°26	5328	52°28	5124
20							26°55 5726								
21	24°39	25°10	2543	26°18	26°53	27°30	28 10	28 30	29°82	30°17	31°4	3182	32°43	3386	3401
22							56°14								
22	59 20	58°41	58°1	57°18	56°36	55°51	55 4	54°17	53°26	52°32	51°28	5041	4941	4837	4732
23							35°85 53°87								
24	27°57	2831	29°7	2943	30°22	31,5	31°46	32°28	33°14	34°ı	34°50	3542	36°35	3780	38°26
25	57:5	56°35	3092	55 11 30°49	54°26	5344	52°61	52°2	51 10	50°6	49 20 36°0	49°22	4721	46°18	45 12 3941
25	56°15	5584	54°52	54°9	53 23	52°36	51°48	50057	50°s	49°11	48°14	4710	46015	45°11	44°5
26							33°58 50°46								
27	31°3	31 39	32°18	32°57	3337	34°20	35°3	3549	3631	3725	380,6	39 %	404	4101	420
20	32°2	53 37 32°39	52 64 33°18	3357	3439	3521	49 <sup>2</sup> 45	36 62	48 2 3740	477 38 20	39%	45 10	44°10 41°9		-
40	- A						48 47						459	42°7	
29							37°8 47°50						4213		
30	3387	3436	35°15	3586	36°38	3721	38°7 46°55	3853	3943	4031	4125	423			
31	34°53	3531	3611	36 52	37°35	3820	39°5	3952	4041	418					
							46° 1				4625				
32	400	400	400	4-0	100	4.0	0	4.0	4.0	42 2B					
33	36°09	3799	38%	3842	3926	40°0	40°56 44°18	4144	4233						
34	3732	38°11	38°53	39°35	409a	41°4	4149			1					
24	-70 12	4727	46°43	45°57	45 8	44 20	4329	7 2 3/							
35	4724	46° 39	45°54	45°8	44°20	43 81		,							
36	1			41 15		4245									
27	40°0	4040	4122	425											
0/		-		4337											
38	45°7	4424	4339	4252											
39		420	4200												
40	43°10	. 0													
H	43°2														
41	432														
			10												

## TABLE 3.—(Continued.)

#### ANGLE OF FACE.

GEAR.



$$g_a = 90^{\circ} - (\alpha_a + \beta)$$
  

$$g_b = 90^{\circ} - (\alpha_b + \beta)$$

(See page 13.)

# TABLE 4 Angle of Face.—Gear.

		72	71	70	69	68	67	66	65	64	63	62	61	60	59	58	57
İ	12	7=°53°	8°	8°-7'	8° 14'	8° 21' 78° 19'	8°28	8°35'	8° 43	8° 51	8° 59'	9° 7'	9" 17	9°26	9° 35	9° 45	9°55'
ł	13	8°40	8°48	8° 54'	3. S.	9. 9.	9° 18'	9° 26	9° 35	9°43	9° 52	10, 1,	10, 11,	10° 21	10°31	10°42	10°53
ŀ	IA	90,26	9°34	9°42	9° 50	77°31 9°59	10 8	10°16	10.52	10:35	10°45	10°54	11° 5′	11, 16	11°27	11°39	11.50
+	-					76 43 10°47		76 18 11° 6'							74 45		
-	-					75°55 11°37											
-	10	75 ss	75 43	75 31	75 20	75 7 12°24	74 54	74 40	74 27	74°13	<b>73°</b> 59	73°44	73°28	73°13′	72° 56	72°40	72 23
1	17	75 10	74 58	74 46	74°33	74°20	74° 5	73°52	73 38	73°23	73° 9	72 52	72 35	72°21	72° 3′	71°45	71°28
	18	74 25	12°40 74°12	74°	73 46	13°12' 73°32	73°19′	73° 4'	7249	72 33	72'18	72° 2'	71°44	71°28	71°10	70°51	70 34
	19			13° 36 73° 14	13°48 73°	14° 72°46		14° 24 72° 16									
	20			14° 23 72° 29		14°46 72°		15° 11' 71° 29									
ı	21	14°43	14°55	15° 8'	15°21	15°33′ 71°13′	15°46	15, 23	16°13	16° 28	16°42	16°58	17913	17°28	17°46	18° 2'	18°30
	22	15° 27	15°40	15° 53	16.6	16° 20	16° 33	16° 47	17 2	17°16'	17°31	17°49	16" 3"	18° 20	18° 37	18° 56	19'13
	23	16' 12	16°24	16°38	16° 51'	70°28 17°5	17:20	17°34	17°50	18° 5'	18"26	18°36	18°54	19° 10	19°28	19°48	20'5
		16° 55	17° 9	17°22	17°37	69 43 17°51	18.6.	18°81	18° 37	18. 23	19° 9'	19°26	19'44	20° 1	2019	20,39	20'58
	24					68°59 18°36′											
	<u>25</u>	69°21	69°4	68°48	68°31	68°14 19°22	67°56	67°37	67°18	67°	66 35	66°20	65°58	65°37	65 14	64°51 22°26	64 28
	<u> 26</u>	68 39	68°22	68° 5	67 48	67°30	67°13	66 53	65°34	66°14	65°53	65°32	65 11	64 49	64 26	64°2	63 39
	27	67 57	67 39	67°22	67 5	20° 6' 66°46	85 88	66° 8'	65 48	65°29	65° 8'	64 46	64 24	64° 1	63°39'	63° 14	62 49
	28	19'46	20°1'	20°17	20°32 66°22	20°50 66°4	21°6' 65°44	21°23	21°41 65°5	22°	22° 18′ 64° 22′	22°37 64° 1'	22°56′ 63°38′	23°17 63°15′	23°38 62°52	23°69′ 62°27′	24°21 62°1
	29	20 27	20°43	SO 29	21°16	21°33' 63°21	21.20	55°8	22°27	22°45	23° 5′	23°25	23°44	24'4	24°25	24 48	25'10
ı	30	S1.8	21°25	21,42	21,28	22°15 64°39	22,34	22,25	23.10	23°30	23° 50'	24"10	24°30	24 51	25'12	25°3€	25,23
1	31	21°50	25°6	22°24	22.41	22°59'	23°17	23°35	23'55	24°14	24°34	24°54	25°17	25°38	25°58	26'22	26'46
1	22	22,31	22°48	23'4	23°23	63°57 23°40	23° 59'	24°18′	24 38	24° 58'	25,18,	25°39	26°1'	26°23	26°45	27'9'	2734
1	32					63°.16' 24° 22											
		63 56	63°36	63° 16	62°56	62°35	62 15	61°53	61°31	61°8′	60 44	60 20	59 55	59°31′	59 5	58 38	58° 11
	34																
-	35	62 39	62 18	61,28	61 37	61 16	60 54	60 32	60 9	59 44	59 22	58°56	58 31	58°6	57 39	57 11	5644
	30	62" 1	61 41	61,50	50 59	26°24 60°36	60 15	5951	5928	59 4	58 40	58 15	5750	57 24	56 57	56°29	56° 1'
	37	61 23	86°6	26°25	26°44 60°20	27°4′ 59 58	27°25 59 35	27°45 59°13	28°7	28°29 58°25	28°51'	29°15'	29°38	30° 2	30°27	30°52 55°48	31°18
		10.0	0.00	A-0 4		27°44 59°20	-	-			20.0		-	-	-		
1	39	27° 3	27°22	27°42	28° 2'	28°22 58°42	28°43	29° 5′	29°27	29 49	30°13	30°36	31°1′	31, 56	31.50	32°16	32°43
-	40	E1-40	140	59.50	28 41	59.1	33. SS	2943	30 6	30,58	307.52	31 17	31 42	32 6	35,31	32 57	33 24
-	41	28°17	28 37	28'57	29" 18	58° 5 29° 39′	30,1,	30°zź	30°45	31, 9,	31°∋í	31°55'	32,50	32,46	33°12	33,32	34 6
		Drawn	0.0	0.00	man	57 29 30°16	2 2 2 2			CONTRACTOR OF THE PARTY.			200	300 /	100		Contraction in
1	42	58 22	38° 1'	57 39	57° 15'	30°16 56°52	56° 28	56° 4'	55°39	55 13'	54°48	54 21	53 54	53 26	52 58	52 29	52°

ZCIZI

	56	55	54	53	52	51	50	49	48	47	46	45	44	43	42
12	10, 6,	10° 16'	10°28	10° 39'	10°52	II° 3′	(1° 15	11° 30	II°43	11° 58′	15, 13	12, 53	12°45	13° 1′	13°19
16	75 54	75 40 11° 16	75 24 11°28'	75 9'	74 52 11° 54	74 37 12° 8'	74 15 12°20	73 58 12°37	73 39 12° 51	73 20 13° 7	72° 59 13° 23	72 37 13°40	72 15 13°58		71°25
13		74 40	74 zá	74° 8′	73 50	73°32	73° 12	72 53	72 33	72 11	71 49	71°26	71° 2'	70° 38	70° 1 i
14			73 25	73 7	72 49	72° 29	72 8	71 48	71 27	71° 5	14° 33 70° 41	7017	69 52	69 26	68°59
15	13° 1'	13°16	13° 26 72° 26	13°43'	13°59 71°49	14'14	14°30	14'47	15° 5′ 70° 23	15° 23 69 50	15°42 69°34	16° 1′	16° 22 68°42	16°43	17°5′ 6747
16	13°59	14°13	14 28	14°44	15°1'	15°17	15° 35'	15° 52	16"11"	16°30	16° 50	17°10	17°32	17°56	18, 18,
17	14°57	15°11	15°23	15°44	16° 1'	16°18	16° 37	16.22	17'15	17°36	68° 28 17°57	18,50	18°43	19° 6'	19°31
											67°23′ 19°3′				
18	70 14	69°53	69°34	69° 12	68°49	68°26	68° 3′	67 38	67° 12	66 47	66 19	65° 5i	65°26	64°50	64 18
19	69 19	68 58	17°23 68°37	68 15	6752	67° 29	67°4	66 37	66°12	65 44	20° 8′ 65° 16	64 46	64 15	63 44	63 10
20	17°44	18° 1'	18, 19	18°40	19°	19°20	19941	50°5	20°25	20°49	21° 13′ 64° 13	21°39	22°5	22°32	23°
21	18° 39	18°57	19 16	19°37	19'58	20 19	20 41	21° 3'	21°27	21°52	22017	2243	53°10	23°38	24° 8'
_											63°13 23°19				
											62° 11' 24° 21				
23	65 47	65 23	6458	64 33	6 <del>4</del> 8	634i	53 13	62°44	62°15	61 44	61° 13	60°4i	60° 6'	59° 31	58 54
24	21° 19	21°39 64°31	22°1' 64° 5'	22 24 63 46	22°46 63°14	23 10 62 46	23°36 62°19	61 48	61 18	24° 53	25°21 60°15	25 49 59 41	26°20	26°51 58°31	27°23 57 53
25	22°11′	22°33	22°56	23°18′	23°41	24° 7	24° 32	24°57	25°24	25°52	26°20 59°18	26°50	27°21	27°52	58.56
26	23° 3′	23°25	23°47	24°13	24°26	25° 1′	25°28	25°53	26°21	26 49	27°19	2749	15 8S	28°54	29°27
07											58° 21				
213	62 25	6i°58	61 32	61 5	60 37	60°7	59° 38	59 5	58 33	58°	57° 26	56 si	56 15	55 3 <b>8</b>	54 59
28	61°36	61° 9'	60 43	60 14	5946	59 16	58°45	58 13	57 42	57 8	56° 32	55°57	55°20	54 42	54'8
29											30° 8' 55° 46				
30	26° 22	26°47	27°12	27°38	28°4′	28°32	29°	29.58	29°58	30°30	31°2'	31°34	32° 8′	32"44	33.18
31	27°10	27°34	28°3′	28.54	28°54	29,53	29°51	30°20	30°52	31°22	31°55	32°29	33°2'	33°39	34 15
31											53 57 32°46				
32	58 28	57,59	57 31	รว์ เ	56 41	56°	55 28	54 54	54 20	53 44	53 8' 33°38	52 31	51 52	51 13	50 32
33	5743	57 14	56 45	56 15	55 49	5รึ เร	54° 39	54° 5	33° 32	58 38	52° 20	51 42	51° 3'	50 22	49 41
34											34°26 51°32				
35	30°15	30 42	31° 10	31°38	327	32°36	33°7	33°38	34°11	34°4	35017	35°5	36°27	37°5	37°42
36	31°	31°27	31°55	32°23	32 53	33°23	33° 53	34° 2	34°57	35°31	50°45	36041	37°16	37°5	38°32
	55 32 31°45	32°12	54 33 32°40	54 l'	53°28	52° 57	52° 23	51°44	35°43	50 37	49 59	49 21 37°2	48°42	48°1	47 20 39°20
37	54 49	54 20	53 50	53 18	32°46	52 13	51°40	51°4	50° 2	49 5	49 15	48 37	4756	47 16	46 34
38	54 9	53 38	53 8	52 38	52 4	51°30	50 56	50° 2	1494	49 9	37°38 48°32	4752	47 13	46 32	4351
39	33°10'	33°35	34° 7	34°36	35°7	35°37	36°9	36°4	37°15	37°4	38°24	39°	39 36	40°13	403
40	33°52'	34°21	34"50	35°18	35°49	36°2	36°53	37°2	37°51	38°3	39 8	39°44	40°20	4058	41°37
	34°33	35° 3'	35°31	36° 1	36°31	37° 3	37°35	38° 7	138*4	1 39 1	47° 6'	40°2	41° 5	4104	42 22
41		Dec. 0				0	. /2	0 10	A OA	. 0	46° 25 8 40° 34				
42	51°36	50 59	50°28	49 54	49 2	48 41	48 13	47°3	47	462	45° 46	45° 7	442	43°4	43 4

INION.

# NATURAL SINE.

	1							li I
Deg.	0'	10'	20'	30′	40′	50′	60′	
0	.00000	.00291	.00581	.00872	.01163	.01454	.01745	89
1	.01745	.02036	.02326	.02617	.02908	.03199	.03489	88
2	.03489	.03780	.04071	.04361	.04652	.04943	.05233	87
3	.05233	.05524	.05814	.06104	.06395	.06685	.06975	86
4	.06975	.07265	.07555	.07845	.08135	.08425	.08715	85
5	.08715	.09005	.09295	.09584	.09874	.10163	.10452	84
6	.10452	.10742	.11031	.11320	.11609	.11898	.12186	83
7	.12186	.12475	.12764	.13052	.13341	.13629	.13917	82
8	.13917	.14205	.14493	.14780	.15068	.15356	.15643	81
9	.15643	.15930	.16217	.16504	.16791	.17078	.17364	80
10	.17364	.17651	17937	.18223	.18509	.18795	.19080	79
11	.19080	.19366	.19651	.19936	.20221	.20506	.20791	78
12	.20791	.21075	.21359	.21644	.21927	.22211	.22495	77
13	22495	.22778	.23061	.23344	.23627	.23909	.24192	76
14	.24192	.24474	24756	.25038	.25319	.25600	.25881	75
15	.25881	26162	26443	.26723	.27004	.27284	.27563	74
16	.27563	.27843	28122	.28401	.28680	.28958	.29237	73
17	.29237	.29515	. 29793	.30070	.30347	.30624	.30901	72
18	.30901	.31178	.31454	.31730	.32006	.32281	.32556	71
19	.32556	.32831	.33106	.33380	.33654	.33928	.34202	70
20	.34202	.34475	34748	.35020	35293	.35565	.35836	69
21	.35836	.36108	.36379	.36650	.36920	.37190	.37460	68
22	.37460	.37730	.37999	.38268	.38536	.38805	. 39073	67
23	.39073	.39340	.39607	.39874	.40141	.40407	.40673	66
24	.40673	.40989	.41204	.41469	.41733	.41998	.42261	65
25	.42261	.42525	.42788	.43051	.43313	.43575	.43837	64
26	.43837	. 44098	.44359	. 44619	.44879	.45139	.45399	63
27	.45399	.45658	.45916	.46174	. 46432	.46690	.46947	62
28	.46947	.47203	.47460	.47715	.47971	.48226	.48481	61
29	.48481	.48735	.48989	.49242	.49495	. 49747	.50000	60
30	.50000	.50251	.50503	.50753	.51004	.51254	.51503	59
31	. 51503	.51752	. 52001	.52249	. 52497	. 52745	. 52991	58
32	.52991	. 53238	.53484	.53730	. 53975	.54219	. 54463	57
33	. 54463	.54707	.54950	. 55193	.55436	.55677	. 55919	56
34	. 55919	. 56160	.56400	. 56640	. 56880	.57119	. 57357	55
35	.57357	.57595	.57833	.58070	.58306	.58542	.58778	54
36	.58778	.59013	.59248	.59482	.59715	.59948	.60181	53
37	.60181	.60413	.60645	.60876	.61106	.61336	.61566	59
38	.61566	.61795	.62023	. 62251	.62478	.62705	.62932	51
39	.62932	.63157	. 63383	.63607	.63832	.64055	. 64278	50
40	.64278	.64501	.64723	.64944	.65165	.65386	.65605	49
41	.65605	.65825	.66043	.66262	. 66479	. 66696	.66913	48
42	.66913	.67128	.67344	.67559	.67773	.67986	.68199	47
43	.68199	.68412	.68624	.68835	.69046	. 69256	.69465	46
44	. 69465	.69674	.69883	.70090	.70298	.70504	.70710	45
	60′	50′	40′	30′	20′	10′	0'	Deg.
1								

NATURAL COSINE.

# NATURAL SINE.

						*		
Deg.	0′	10'	20′	30′	40'	50′	60′	
45	.70710	.70916	.71120	.71325	.71528	.71731	.71934	44
46	.71934	.72135	.72336	.72537	.72737	72936	.73135	43
47	.73135	.73333	73530	.73727	.73923	.74119	74314	42
48	.74314	.74508	.74702	.74895	.75088	.75279	75471	41
49	.75471	75661	.75851	.76040	.76229	.76417	76604	40
50	76604	.76791	. 76977	.77162	.77347	.77531	77714	39
51	77714	.77897	.78079	.78260	.78441	.78621	.78801	38
53	.78801	.78979	.79157	.79335	.79512	.79688	.79863	37
53	79863	.80038	.80212	.80385	.80558	.80730	.80901	36
54	.80901	.81072	.81242	.81411	.81580	.81748	.81915	35
55	.81915	.82081	.82247	.82412	.82577	.82740	.82903	34
56	.82903	.83066	.83227	.83388	.83548	.83708	.83867	33
57	.83867	.84025	.84182	.84339	.84495	.84650	.84804	32
58	.84804	.84958	.85111	.85264	.85415	.85566	.85716	31
59	.85716	.85866	.86014	.86162	.86310	.86456	.86602	30
60	.86602	.86747	.86892	.87035	.87178	.87320	.87462	29
61	.87462	.87602	.87742	.87881	.88020	.88157	.88294	28
62	.88294	.88430	.88566	.88701	.88835	.88968	.89100	27
63	.89100	.89232	.89363	.89493	.89622	.89751	.89879	26
64	.89879	.90006	.90132	.90258	.90383	.90507	.90630	25
$6\overline{5}$	.90630	.90753	.90875	.90996	.91116	.91235	.91354	24
66	.91354	.91472	.91589	.91706	.91821	. 91936	.92050	23
67	.92050	.92163	.92276	.92388	92498	.92609	.92718	22
68	.92718	.92827	.92934	.93041	.93148	.93253	. 93358	21
69	.93358	.93461	. 93565	.93667	.93768	.93869	.93969	20
70	. 93969	.94068	.94166	.94264	.94360	.94456	.94551	19
71	.94551	.94646	.94739	.94832	.94924	.95015	.95105	18
72	.95105	.95195	.95283	.95371	.95458	.95545	.95630	17
73	.95630	.95715	.95799	.95882	.95964	.96045	.96126	$\overline{16}$
74	.96126	.96205	.96284	.96363	.96440	.96516	.96592	15
75	.96592	.96667	.96741	.96814	.96887	.96958	.97029	14
76	.97029	.97099	.97168	.97237	.97304	.97371	.97437	13
77	.97437	.97502	.97566	.97629	.97692	.97753	.97814	12
78	.97814	.97874	.97934	.97992	.98050	.98106	.98162	11
79	.98162	.98217	.98272	.98325	.98378	.98429	.98480	10
80	.98480	.98530	.98580	. 98628	.98676	.98722	.98768	9
81	. 98768	.98813	.98858	.98901	.98944	.98985	.99026	8
82	.99026	. 99066	.09106	.99144	.99182	.99218	.99254	7
83	. 99254	.99289	.99323	.99357	.99389	.99421	.99452	6
84	. 99452	.99482	.99511	.99539	.99567	.99593	.99619	5 i
85	.99619	.99644	.99668	.99691	.99714	.99735	.99756	4
86	.99756	.99776	.99795	99813	.99830	.99847	.99863	3
87	. 99863	.99877	.99891	.99904	.99917	.99928	.99939	2
88	. 99939	.99948	.99957	.99965	.99972	.99979	.99984	1
89	.99984	.99989	.99993	.99996	.99998	.99999	1.0000	0
-	60'	50/	40'	30′	20'	10'	0'	Deg.
		.		- 1			1	J 08.

NATURAL COSINE.

# NATURAL TANGENT.

1								
Deg.	0'	10′	20′	30′	40'	50′	60′	
0	.00000	.00290	.00581	.00872	.01163	.01454	.01745	89
1	.01745	.02036	.00361 $.02327$	.02618	.02909	.03200	.03492	88
2	.03492	.03783	.04074	.02016	.02303	.04949	.05240	87
3	.05240	.05532	.05824	.04300	.06408	.06700	.06992	86
4	.06992	.07285	.03624 .07577	.07870	.08162	.08455	.08748	85
5	.08748	.09042	.01311 $.09335$	.09628	.09922			
. 6	.10510	.10804	.11099	.11393	.11688	.10216	.10510	84
7	.12278	.10004 $.12573$	.12869	.11595 $.13165$	.13461		.12278	83
8		$0.12375 \\ 0.14350$	.12609	$.15105 \\ .14945$	.15401 $.15242$	.13757	.14054	82
	.14054					.15540	.15838	81
9	.15838	.16126	.16435	.16734	.17033	.17332	.17632	80
10	.17632	.17932	.18233	.18533	.18834	.19136	.19438	79
11	.19438	.19740	.20042	.20345	.20648	.20951	.21255	78
12	.21255	.21559	.21864	.22169	.22474	.22780	. 23086	77
13	.23086	.23393	.23700	.24007	.24315	.24624	.24932	76
14	.24932	.25242	. 25551	.25861	.26172	.26483	.26794	75
15	.26794	.27106	.27419	.27732	.28046	.28360	.28674	74
16	.28674	.28989	. 29305	.29621	.29938	. 30255	.30573	73
17	.30573	.30891	.31210	. 31529	.31850	.32170	.32492	72
18	.32492	. 32813	. 33136	. 33459	.33783	.34107	.34432	71
19	.34432	.34758	.35084	.35411	.35739	.36067	.36397	70
20	.36397	.36726	.37057	.37388	.37720	.38053	. 38386	69
21	.38386	.38720	. 39055	. 39391	.39727	.40064	. 40402	68
22	.40402	.40741	.41080	.41421	.41762	.42104	.42447	67
23	.42447	.42791	. 43135	.43481	.43827	.44174	. 44522	66
24	.44522	.44871	.45221	.45572	.45924	.46277	. 46630	65
25	. 46630	.46985	.47341	.47697	.48055	.48413	.48773	64
26	.48773	.49133	.49495	.49858	.50221	.50586	.50952	63
27	.50952	.51319	.51687	.52056	.52427	.52798	.53170	62
28	.53170	.53544	. 53919	.54295	.54672	.55051	.55430	61
29	.55430	. 55811	. 56193	.56577	.56961	.57347	.57735	60
30	.57785	.58123	.58513	.58904	.59297	. 59690	.60086	59
31	.60086	.60482	.60880	.61280	.61680	.62083	. 62486	58
32	. 62486	.62892	. 63298	.63707	. 64116	.64528	. 64940	57
33	.64940	. 65355	.65771	.66188	.66607	.67028	.67450	56
34	.67450	.67874	.68300	.68728	.69157	.69588	.70020	55
35	.70020	.70455	.70891	.71329	.71769	.72210	.72654	54
36	.72654	.73099	. 73546	. 73996	.74447	.74900	. 75355	53
37	. 75355	.75812	.76271	.76732	.77195	.77661	.78128	52
38	.78128	. 78598	.79069	. 79543	.80019	.80497	80978	51
39	.80978	. 81461	.81946	.82433	.82923	.83415	. 83910	50
40	.83910	.84406	.84906	.85408	.85912	.86419	. 86928	49
41	.86928	.87440	87955	.88472	.88992	.89515	.90040	48
42	.90040	.90568	. 91099	.91633	. 92169	.92709	.93251	47
43	.93251	.93796	.94345	. 94896	. 95450	.96008	.96568	46
44	.96568	.97132	.97699	.98269	.98843	.99419	1.0000	45
	: 60'	50'	40'	30'	20'	10'	0'	Deg.
-	1							

NATURAL COTANGENT.

## NATURAL TANGENT.

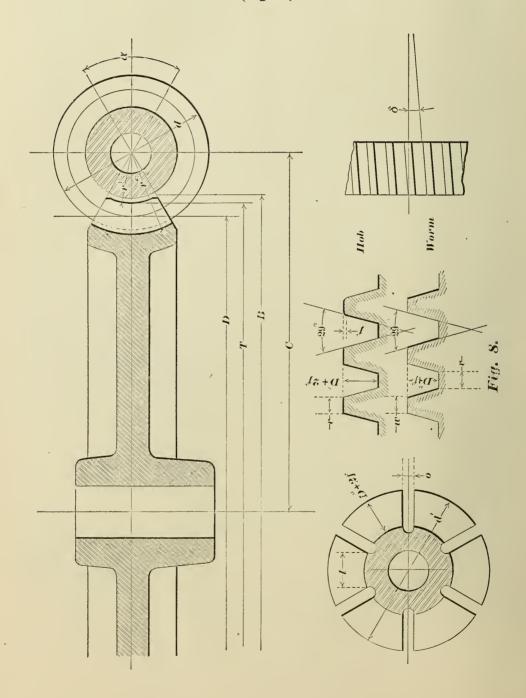
				•				
Deg.	0'	10'	20'	30'	40'	50′	60	
45	1.0000	1.0058	1.0117	1.0176	1.0235	1.0295	1.0355	44
46	1.0355	1.0415	1.0476	1.0537	1.0599	1.0661	1.0723	43
47	1.0723	1.0786	1.0849	1.0913	1.0977	1.1041	1.1106	42
48	1.1106	1.1171	1.1236	1.1302	1.1369	1.1436	1.1503	41
49	1.1503	1.1571	1.1639	1.1708	1.1777	1.1847	1.1917	40
50	1.1917	1.1988	1.2059	1.2131	1.2203	1.2275	1 2349	39
51	1.2349	1.2422	1.2496	1.2571	1.2647	1.2723	1.2799	38
52	1.2799	1.2876	1.2954	1.3032	1.3111	1.3190	1.3270	37
53	1.3270	1.3351	1.3432	1.3514	1.3596	1.3680	1.3763	36
54	1.3763	1.3848	1.3933	1.4019	1.4106	1.4193	1.4281	35
55	1.4281	1.4370	1.4459	1.4550	1.4641	1.4733	1.4825	34
56	1.4825	1.4919	1.5013	1.5108	1.5204	1.5301	1.5398	33
57	1.5398	1.5497	1.5596	1.5696	1.5798	1.5900	1.6003	32
58	1.6003	1.6107	1.6212	1.6318	1.6425	1.6533	1.6642	31
59	1.6642	1.6753	1.6864	1.6976	1.7090	1.7204	1.7320	30
60	1.7320	1.7437	1.7555	1.7674	1.7795	1.7917	$1.8040 \\ 1.8807$	29 28
61	1.8040	$1.8164 \\ 1.8940$	1.8290 $1.9074$	1.8417 $1.9209$	1.8546 $1.9347$	1.8676 $1.9485$	1.8607 $1.9626$	27
$\begin{array}{ c c } \hline 62 \\ \hline 63 \\ \end{array}$	1.9626	1.8940 $1.9768$	1.9074 $1.9911$	$\frac{1.9209}{2.0056}$	2.0203	2.0352	2.0503	$\parallel \tilde{26} \parallel$
$\begin{vmatrix} 65 \\ 64 \end{vmatrix}$	2.0503	2.0655	2.0809	2.0050 $2.0965$	2.0205 $2.1123$	2.0302 $2.1283$	2.0303 $2.1445$	$\begin{vmatrix} z_0 \\ 25 \end{vmatrix}$
65	2.1445	2.0000	2.1774	2.1943	2.2113	2.1205 $2.2285$	2.2460	24
$\begin{vmatrix} 66 \end{vmatrix}$	2.2460	2.2637	2.2816	2.2998	2.3182	2.3369	2.3558	23
67	2.3558	2.3750	2.3944	2.4142	2.4342	2.4545	2.4750	$\begin{vmatrix} 20 \\ 22 \end{vmatrix}$
68	2.4750	2.4959	2.5171	2.5386	2.5604	2.5826	2.6050	21
$\begin{vmatrix} 69 \end{vmatrix}$	2.6050	2.6279	2.6510	2.6746	2.6985	2.7228	2.7474	$\begin{vmatrix} \tilde{20} \end{vmatrix}$
70	2.7474	2.7725	2.7980	2.8239	2.8502	2.8770	2.9042	$1\tilde{9}$
71	2.9042	2.9318	2.9600	2.9886	3.0178	3.0474	3.0776	18
72	3.0776	3.1084	3.1397	3.1715	3.2040	3.2371	3.2708	17
73	3.2708	3.3052	3.3402	3.3759	3.4123	3.4495	3.4874	16
74	3.4874	3.5260	3.5655	3.6058	3.6470	3.6890	3.7320	15
75	3.7320	3.7759	3.8208	3.8667	3.9136	3.9616	4.0107	14
76	4.0107	4.0610	4.1125	4.1653	4.2193	4.2747	4.3314	13
77	4.3314	4.3896	4.4494	4.5107	4.5736	4.6382	4.7046	12
78	4.7046	4.7728	4.8430	4.9151	4.9894	5.0658	5.1445	11
79	5.1445	5.2256	5.3092	5.3955	5.4845	5.5763	5.6712	10
80	5.6712	5.7693	5.8708	5.9757	6.0844	6.1970	6.3137	$\parallel 9 \parallel$
81	6.3137	6.4348	6.5605	6.6911	6.8269	6.9682	7.1153	8
82	7.1153	7.2687	7.4287	7.5957	7.7703	7.9530	8.1443	7
83	8.1443	8.3449	8.5555	8.7768	9.0098	9.2553	9.5143	$  \begin{array}{c c} 6 \end{array}  $
84	9.5143	9.7881	10.078	10.385	10.711	11.059	11.430	5
85	11.430	11.826	12.250	12.706	13.196	13.726	14.300	4
86	14.300	14.924	15.604	16.349	17.169	18.075	19.081	3 2
87	19.081 28.636	$\begin{vmatrix} 20.205 \\ 31.241 \end{vmatrix}$	$21.470 \\ 34.367$	$22.904 \\ 38.188$	$\begin{vmatrix} 24.541 \\ 42.964 \end{vmatrix}$	$\begin{vmatrix} 26.431 \\ 49.103 \end{vmatrix}$	$\begin{bmatrix} 28.636 \\ 57.290 \end{bmatrix}$	$\begin{bmatrix} z \\ 1 \end{bmatrix}$
89	$\frac{28.050}{57.290}$	68.750	85.939	114.58	171.88	$\frac{49.105}{343.77}$	57.290 ∞	$\begin{vmatrix} 1 \\ 0 \end{vmatrix}$
00	1 01.200	00.100	00.000	114.00	171.00	040.11	$\omega$	
	60'	50	40'	30′	20'	10'	0*	Deg.

NATURAL COTANGENT.

## CHAPTER IV.

# WORM AND WORM WHEEL.

(Fig. 8.)



#### FORMULAS.

L = lead of worm.

N = number of teeth in gear.

m =threads or turns per inch in worm.

d = diameter of worm.

d' = diameter of hob.

T = throat diameter.

B = blank diameter (to sharp corners).

C = distance between centers.

o = thickness of hob-slotting cutter.

l =width of lands at bottom.

b = pitch circumference of worm.

v =width of worm thread tool at end.

w =width of worm thread at top.

P = diametral pitch.

P' = circular pitch.

s = addendum and module.

t =thickness of tooth at pitch line.

 $t^n$  = normal thickness of tooth.

f = clearance at bottom of tooth.

D'' =working depth of tooth.

D'' + f = whole depth of tooth.

 $\delta$  = angle of tooth of worm wheel with its axis, or the angle of thread of worm with a line at right angles to its axis.

If the lead is for single, double, triple, etc., thread, then

$$L = P'$$
, 2 P', 3 P', etc.

$$\alpha = 60^{\circ} \text{ to } 90^{\circ}$$

$$L = \frac{I}{m}$$

$$P' = \frac{\pi T}{N+2}$$

$$D = \frac{N P'}{\pi} = \frac{N}{P}$$

$$T = \frac{N}{P} + 2 s$$

$$b = \pi (d-2 s)$$

$$\tan \delta = \frac{L}{b} \quad \begin{cases} \text{Practical only when width of wheel on wheel pitch circle is not more than } \frac{1}{2} \text{ pitch diameter of worm.} \end{cases}$$

$$t^{n} = t \cos \delta$$

$$t^{1} = \frac{d}{2} - 2 s$$

$$t^{2} = t^{1} + D'' + f$$

$$t^{2} = \frac{D + d}{2} - s$$

$$t^{3} = \frac{1}{2} - s$$

$$t^{3} = \frac{1}{2} + \frac{$$

Note.—The notations and formulas referring to tooth parts, given on page 5 for spur gears, apply to worm wheels, and are here used.

Note.—Hob and worm should be marked, as per example:

4 turns per 1" single .25 P; .25 L.

2 turns per 1" double .25 l"; .50 L.

## UNDERCUT IN WORM WHEELS.

In worm wheels of less than 30 teeth the thread of the worm (being 29°) interferes with the flank of the gear tooth. Such a wheel finished with a hob will have its teeth undercut. To avoid this interference two methods may be employed.

First Method. - Make throat diameter of wheel

$$T = \cos^{2} 14 \frac{1}{2} \circ \frac{N}{P} + 4 s \qquad \text{or}$$

$$T = \frac{.937 \text{ N}}{P} + 4 s$$

This formula increases the throat diameter, and consequently the center distance. The amount of the increase can be found by comparing this value of T with the one as obtained by formula on page 36. To keep the original center distance, the outside diameter of the worm must be reduced by the same amount the throat diameter is increased.

Second Method.—Without changing any of the dimensions we found by the formulas given on page 36, we can avoid the interference to be found in worm wheels of less than 30 teeth by simply increasing the angle of worm thread. We find the value of this angle by the following formula:

Let there be

 $2 \gamma = \text{angle of worm thread.}$ N = number of teeth in worm wheel.

$$\cos \gamma = \sqrt{1 - \frac{2}{N}}$$

From this formula we obtain the following values:

As this latter formula involves the making of new hobs in many cases, on account of change of angle, we prefer to reduce the diameter of worm as indicated by first method, if the distance of centers must be absolute.

TABLE OF ANGLES FOR GASHING WORM WHEELS.—SINGLE THREADED.

2000	-1::							-					1				
0003.00	= 01												-	-			
00.2.00												,	,				
331.50	93 99													,			
001.33	s =							<u> </u>				-	0	1			
00,1,00							-		-								,,
0, 800	= ::						1							12		1, 0-1,	7' 5-49'
6'.750	==											`.		0-26	3, 6-3,	5-44	6-2
999",	=   01						1			<u> </u>		6-40 6-27	,6-3	5-42	4-51 6-23	, 5-0,	4-51
,,000(	01 00 ped									, e-45,	6-14		6-27	,0-9,	4-51	4-30	4-22
.6714	e=								6-55	4-20 4-48 4-60 6-36 6-23	6-69	5-32	4-33 5-12	3-13 3-26 3-40 3-40 4-17 4-54 6-8	4-37	3-17' 3-26' 3-60' 4-23' 4-36' 5-6'	4-10
,5000	91						7-15	6-36	6-3	5-36	5-12	10-10-1	4-33	4-17	1-3	3-60	3-30
.4444	61				8-3,	7-10	6-27	6-52	6-23	4-69	4-37	4-10	4-3	3-40	3^28' 3-36'	3-26	3-14
.4286	@1 - :::		9-38' 10-18'	8-52	7-47	6-55	0214	5-40	5212	4-48	4-27	4-10	3-54	3-40	3 28	3-17	3-7,
.4000	- 21 - 21	10-30/10-49/11-31/	9-38,	8-17	0-36   0-48   7-16   7-47	0-27	5-40' 6-14' 6-27' 7-15'	2-12	4-25 4-33 4-51 5:12	4-20	4210	3-63	3-10 3-25 3-30 3-54 4-3	3-26	3-14	3-4	2-65
.3750	91 31 33	0-40	0-3	7-46	0-48	6-4	6-27	4-50	4-33	4-12	3-54	3-30	3-25	3-13	3-2	2-65-	2-44′
.3636	91 m =	10-30	8-46	7-32′	0-36	5-52	4-51' 5-17' 6-27'	4-40	4-25		0-47	3-32	3-10	2-62' 3-7'	2-42 2-57	2-33 2-47 2-55- 3-4'	2-30
,3333	93	)-38,		6-55	6-3	6-23	4-51	4-25	4-3,	3-44	3-20	3-14	3-2/	2-62	2-42	2-33	2-26
.2857	-   00 -   01	7-16 8-17 9-38	6-4 6-55 8-3	6-12' 6-66' 6-65' 7-32' 7-46'		4-37 (6-23 6-52	4-10′	3-10' 3-47- 4-26' 4-40' 4-66' 5-17' 5-40' 5-62' 6-36'	3-28	2-46 3-12 3-44 4-4	2-30 2-58 3-28 3-47 3-54 4-10 4-27 4-37 5-19	2-20' 2-47' 3-14' 3-32' 3-30' 3-53' 4-10' 4-10' 4551' 5-32'	2-36	2-27	2-10	2-12	2-5'
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2222	- ±1	6-28	5-23	4-37	4-3	3-36	3-14	2-57'	2-42	2-30,	2-19	2-10	2-2	-54	1-48' 2-2'	2.42	-37′
2000,"	73	5-49	4-51	1-10	3-30	3-14		2-30	2-20	2-15	2-6	1-67	1-40,	17.43	1-37′	232	1-27
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1538	9 ::	4-20	3-44	3-12	2-48	2-30	2215	2-2	1-52	1-44	1-30,	1-30	1-24	1-10	12.16	1211	1-7
1429	[~	4-10	3-28	2-50,	2-36	2-10	2-6	1-64	1-44′	12.26	1229	1223	1510	1214	1-0,	126	1-3′
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LEAD.	TURNS PER INCH.		1						JWE.								

TABLE OF ANGLES FOR GASHING WORM WHEELS.—SINGLE THREADED.

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0000	-	6-55	6-36′	6-19, 8	6-3, 8	5-36' 7	5-12 6	4-51' 6	4-33 6	4-17 5	4-3' 5	3-50 5	3-39 4	3-28 4	3-19' 4	3-10 4	3-2' 4
.8000″1.0	14	5-32' 6	5-17, 6	5-4' 6	4-51 6	4-29 5	4-10' 5	3~53, 4	3-39, 4-	3-26 4-	3-14 4	3-4,	2-55 3	2-47 3-	2-39, 3	2-32 3-	2-26   3
.7500″.8	1 3	5-12' 5	58,	4-45 5	4-33' 4	4-12' 4-	3-54 4-	3-39, 3-	3-25' 3-	3-13 3	3-2' 3-	2-53' 3-	2-44' 2-	2-36, 2-	2-29, 2-	2-23' 2-	2-17' 2-
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.4444".5	10.	325 3	2-57′3°	2-49   3-	2-42' 3	2-30, 2-	2-19' 2-	2-10' 2-	2-2 2-	1-54 2	1-48 2-	1-42′ 1	1-37′ 1-	1-33, 1-	1-28 1-	1-24':1-	1-21, 1-
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.3750".4	2 2 2 2	2 <sup>2</sup> 36′  2 <sup>2</sup>	2-29, 2-	2-23′ 2-	2 <sup>2</sup> 17' 2 <sup>2</sup>	2-6 2-	1-57'   2-	1249′  1	1-43/ 1-	1-37 12	1-31/ 1-	1-26 1-	1-22, 1-	1-18'-1	1-15, 1-	1-11	1-8' 1-
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.2000″.22	- 19 - 19	23,	1-20 1-	1-10, 1-	1-13' 1-	-1 , 2-1	1-3, 1-	53′ 1º	55' 1-	52, 5	40, 6	46,	44, 4	42, 4	40,4	33, 4	4
.1818".20	-de1	1-16/12	12, 12	1-9, 1-	1-6, 1-	<u>-1</u>	57 1-	53,	50, 5	47,	44, 4	45, 4	40, 4	23, 4	36,		
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#### CHAPTER V.

# SPIRAL OR SCREW GEARING.

(Figs. 9, 10, 11.)

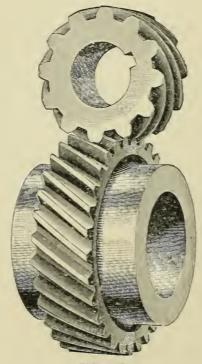


Fig. 9.

RIGHT HAND SPIRAL GEARS.

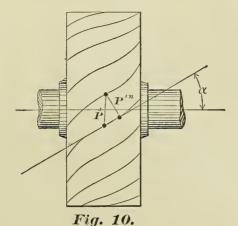
In spiral gearing the wheels have cylindrical pitch surfaces, but the teeth are not parallel to the axis. The line in which the pitch surface intersects the face of a tooth is part of a screw line, or helix, drawn at the pitch surface. A screw wheel may have one or any number of teeth. A one-toothed wheel corresponds to a one-threaded screw, a many-toothed wheel to a many-threaded screw. The axes may be placed at any angle.

Consider spiral gears with:

I. Axes parallel.

II. Axes at right angles.

III. Axes any angle.



LEFT HAND SPIRAL GEAR.

#### Let there be:

 $\begin{bmatrix}
N_a = \\
N_b =
\end{bmatrix}$  number of teeth in gears  $\begin{cases}
a \\
b
\end{cases}$ 

C = center distance.

P = diametral pitch

P' = circular pitch.

 $P^n$  = normal diametral pitch.

 $P'^n = normal circular pitch.$ 

 $\gamma =$  angle of axes.

 $L_1 =$ exact lead of spiral on pitch surface.

 $L_2$  = approximate lead of spiral on pitch surface.

T = number of teeth marked on cutter to be used when teeth are to be cut on milling machine.

D = pitch diameter.

B = blank diameter.

 $\alpha_a = \begin{cases} \alpha_b = \end{cases}$  angle of teeth with axis

t =thickness of tooth.

s = addendum and module.

D'' + f = whole depth of tooth.

Note.—Letters a and b occurring at bottom of notations refer to gears a and b.

#### I.—AXES PARALLEL.

Gears of this class are called twisted gears. The angle of teeth with axes in both gears must be equal and the spirals run in opposite directions. The angles are generally chosen small (seldom over 20°) to avoid excessive end thrust. End thrust may, however, be entirely avoided by combining two pairs of wheels with right and left-hand obliquity. Gears of this class are known as Herringbone gears. They are comparatively noiseless running at high speed.

#### II.—Axes at Right Angles.

Here we must always have:

- 1. The teeth of same hand spiral;
- 2. The normal pitches equal in both gears; and
- 3. The sum of the angles of teeth with axes =  $90^{\circ}$ .

#### CHOOSING ANGLE OF TEETH WITH AXES.

- I. If in a pair of gears the ratio of the number of teeth is equal to the direct ratio of the diameters, i. e., if the number of teeth in the two gears are to each other as their pitch diameters, then the angles of the spirals will be 45° and 45°; for, this condition being fulfilled, the circular pitches of the two gears must be alike, which is only possible with angles of 45°. In such a combination either gear may be the driver.
- 2. If the ratio of the diameters determined upon is larger or smaller than the ratio of the number of teeth, then the angles are:

 $\tan \alpha_a = \frac{D_a N_b}{D_b N_a} \qquad \tan \alpha_b = \frac{D_b N_a}{D_a N_b}$ 

In such gears the velocity ratio is measured by the number of teeth, and not by the diameters.

3. Given Na, Nb and C:

If  $P_{a}$  is made =  $P_{b}$ , then we have case "1" and

$$P' = \frac{\pi C}{\frac{1}{2}(N_a + N_b)}$$

But if  $P_{a}'$  is assumed, then:

$$P_{b}' = \frac{C \pi - \frac{1}{2} N_{a} P_{a}'}{\frac{1}{2} N_{b}}$$

and

$$\tan \alpha_a = \frac{P_a'}{P_b'} \qquad \tan \alpha_b = \frac{P_b'}{P_a'}$$

The gear whose P' or  $\alpha$  is larger will ordinarily be the driver, on account of the greater obliquity of the teeth.

4. Given Na, Nb and C or D.

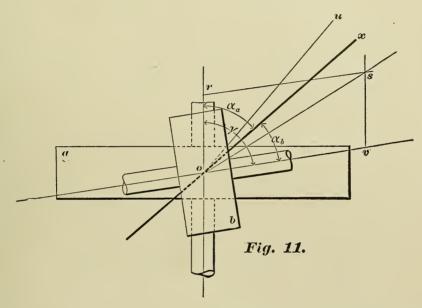
See case "7" under III., considering  $y = 90^{\circ}$ .

# III.—Axis at any Angle $(\gamma)$ .

- 5. Given case "1," under II., then angles of spirals =  $\frac{1}{2}\gamma$ , for the same reason.
- 6. Analogous cases to "2" and "3," under II., may be worked out, when angles of axes  $= \gamma$ , but they have been

omitted, partly because the formulas are too cumbersome, and partly because they are to some extent covered by cases "5" and "7."

7. Given  $N_a$ ,  $N_b$  and C, or one of the pitch diameters. We find the angles by a graphic method, which for all practical purposes is accurate enough; ro and vo are the axes of gears forming angle  $\gamma$  (see diagram, Fig. 11.) On these axes we lay off lines or and ov representing the ratio of the number of teeth (velocity ratio), so that  $N_a: N_b::rs:sv$ , and



construct parallelogram o r s v. Then, according to McCord,\* the angles formed by the tangent s o in the pitch contact o with the axes of the gears insures the least amount of sliding. In bisecting angle  $\gamma$  by tangent u o and using angles produced in this manner we equally distribute the end thrust on both shafts. Both methods have their advantages; to profit by both we select angles  $\alpha_a$  and  $\alpha_b$ , produced by tangent o x, bisecting angle u o s.

Thus we have when angles are found and C given,

$$P'^{n} = \frac{{}^{2} C \pi \cos \alpha_{a} \cos \alpha_{b}^{r}}{N_{a} \cos \alpha_{b} + N_{b} \cos \alpha_{a}}$$

and when  $D_a$  given

$$P'^n = \frac{D_a \pi \cos \alpha_a}{N_a}$$
 and  $D_b = \frac{P'^n N_b}{\pi \cos \alpha_b}$ 

<sup>\*</sup> McCord, Kinematics, page 278.

#### GENERAL FORMULAS.

$$y = \alpha_{a} + \alpha_{b}$$

$$P_{a}^{\prime n} = P_{b}^{\prime n}$$

$$D = \frac{P' N}{\pi} \quad \text{or} = \frac{P'^{n} N}{\pi \cos \alpha}$$

$$B = D + 2s \quad \text{or} = D + \frac{2}{P^{n}}$$

$$P' = \frac{D \pi}{N} \quad \text{or} = \frac{P'^{n}}{\cos \alpha}$$

$$P'^{n} = P' \cos \alpha$$

$$P^{n} = \frac{\pi}{P'^{n}} \quad \text{(Pitch of cutter.)}$$

$$s = \frac{P'^{n}}{\pi} \quad \text{or} = \frac{I}{P^{n}}$$

$$t = \frac{P'^{n}}{2}$$

$$D'' + f = 2s + \frac{t}{Io}$$

$$T = \frac{N}{\cos^{5}\alpha} \quad \text{(See Note 1.)}$$

$$L_{1} = \frac{N P'}{\tan \alpha} \quad \text{or} \quad \frac{N\pi}{P \tan \alpha} \quad \text{or} \quad \frac{L_{1a} = N_{a} P'_{b}}{L_{1b} = N_{b} P'_{a}}$$

$$L_{2} = \frac{Io W G_{2}}{S G_{1}} \quad \text{(See Note 2 and examples.)}$$

$$\begin{pmatrix} \cos^{5} +5^{5} = .3535 \\ \tan +5^{\circ} = 1.000 \end{pmatrix}$$

Note 1.—Cutters of regular involute system.

Use No.	IC	utter	for	T	from	135 u	p.		No.	5	cutter	for	T	from	2	2 I	to	25
	2	à a	5.4	• •	4.6	55 t	0	134	+ 6	6	4.6			4.5	1	7	to	20
6.4	.3	* b	* *	6.0	6.4	35 t	0	54	- 1	7	4.6	4.4	5 6	11	1	14	to	16
6.	4	**	١.	2 6	**	26 t	0	34		S	h b	• •	11		1	12	to	13

Note 2.—Gears used on spiral head and bed for Brown  $\hat{x}$  Sharpe milling machine:

W= number of teeth in gear on worm.  $G_1=$  " " 1st " stud.  $G_2=$  " 2d " stud. S= " " screw.

Should a spiral head of different construction be used, the formula might not apply.

The following data are usually required in cutting spiral gears in a Universal Milling Machine, and it will be found convenient to arrange them in tabular form as follows:

	GEAR.	PINION.
No. of Teeth		
Pitch Diameter		
Outside Diameter		
Circular Pitch		
Angle of Teeth with Axis		
Normal Circular Pitch		
Pitch of Cutter		
Addendum s		
Thickness of Tooth t		
Whole Depth D"+f		
No. of Cutter		
Exact Lead of Spiral		
Approximate Lead of Spiral		
Gears on Milling Machine to Cut Spiral		
Gear on Worm		
ıst Gear on Stud		
2nd Gear on Stud		
Gear on Screw		

If the exact lead  $L_1$  can be obtained by the gears at hand,  $L_1$  will equal  $L_2$  and we shall have from the formula

$$L_{2} = \frac{\text{10 W } G_{2}}{\text{S } G_{1}}$$

$$\frac{L_{1}}{\text{10}} = \frac{\text{W } G_{2}}{\text{S } G_{1}} \text{ (for B. & S. Milling Machine.)}$$

Example I.

Required the gears for cutting a spiral of 2½" lead.

$$\frac{2\frac{1}{2}}{10} = \frac{I}{4} \text{ factoring, in the most simple way, we have}$$

$$\frac{I}{4} = \frac{I \times I}{2 \times 2} = \frac{I \times 28}{56 \times 2} = \frac{32 \times 28}{56 \times 64} = \frac{W G_2}{S G_1}$$

Thus the gearing will be 32 T. on worm, 64 T. 1st. on stud, 28 T. 2nd on stud, and 56 T. on screw.

Trying these gears on the Milling Machine we find that they cannot be used, and as we have no other regular gears in the ratio of 2 to 1 that can be used we must try, by factoring, to get such ratios for the two pairs of gears as to be able to use the gears at hand, bearing in mind that the combined ratio must be  $\frac{1}{4}$ .

$$\frac{1}{4} = \frac{18}{72} = \frac{3 \times 6}{9 \times 8} = \frac{24 \times 6}{9 \times 64} = \frac{24 \times 48}{72 \times 64}$$

These gears are at hand and the combination can be used on the machine, giving the exact lead of 2½".

#### Example II.

Required the gears for cutting a spiral of 8.639" lead.

 $8.639 = 8\frac{639}{1000}$ ; reducing, by continued fractions, to a smaller fraction of approximately the same value, as described on pages 74 and 75

Selecting  $\frac{16}{25}$  as an approximation near enough for our purpose, and in fact as near as we are likely to find gears for, we have for our lead  $8\frac{16}{25}$ . Applying the formula as in Example I.

$$\frac{8\frac{16}{25}}{10} = \frac{W G_2}{S G_1}$$

$$\frac{8\frac{16}{25}}{10} = \frac{216}{250} = \frac{108}{125} \text{ factoring we have}$$

$$\frac{9 \times 12}{25 \times 5} = \frac{9 \times 48}{100 \times 5} = \frac{72 \times 48}{100 \times 40} \text{ the gears required,}$$

these being regular gears furnished with the Milling Machine.

Proof:

$$\frac{72 \times 48 \times 10}{100 \times 40} = 8.640 = L_{2} \\ \underline{8.639} = L_{1} \\ \underline{0.001''} \text{ error in lead.}$$

In shops where much work is done in milling spirals it is desirable to have a full set of gears for the milling machine, from the smallest to the largest numbers of teeth that can be used. This makes it possible, in most cases, to get closer approximations than could be otherwise obtained, and often saves a great deal of figuring.

When the use of continued fractions does not bring a close enough approximation, one method to secure a closer result is to add to or substract from the numerator and denominator of the fraction to be reduced, any numbers nearly in proportion to the given fraction, seeing that the numbers added or substracted are such as to make the fraction reductible to lower terms. By a little ingenuity and patience extremely close approximations can generally be reached in this way.

Take, as an illustration, the fraction in Example II.

$$\frac{8\frac{639}{1000}}{10} = \frac{8639}{10000}$$

Adding 9 to the numerator and 10 to the denominator, these

being in about the same ratio to each other as the numerator and denominator of the fraction, we have

$$\begin{array}{c} 8639 + 9 = 8648 \\ 10000 + 10 = 10010 = \frac{4324}{5005} = \frac{47 \times 92}{55 \times 91} \end{array}$$

All of the gears in this case are special.

Applying the same proof as in Example II. we find that this train of gears will give a lead of 8.6393+, making an error of .0003" in the lead.

No doubt a much closer approximation even than this could be obtained by further trial.

Another method is to multiply both terms of the fraction by some number which will make one term of the fraction easily reducible, and adding one to or subtracting it from the other term to make it possible to reduce that also.

There is an element of uncertainty in both these methods, as we never feel sure that we have obtained the best combination; practical work, however, rarely requires accuracy beyond a point that can readily be reached.

The accompanying list of prime numbers and factors will be found useful in reducing and factoring fractions.

# PRIME NUMBERS AND FACTORS. 1 TO 1000.

1		26	2×13	51	$3 \times 17$	76	$2^2 \times 19$
2		27	$3^3$	52	$2^2 \times 13$	77	7×11
3		28	$2^2 \times 7$	53		78	$2 \times 3 \times 13$
4	$2^2$	29		54	$2 \times 3^3$	79	
5		30	$2 \times 3 \times 5$	55	5×11	80	$2^4 \times 5$
6	$2 \times 3$	31		56	$2^3 \times 7$	81	34
7		32	$2^5$	57	$3 \times 19$	82	$2 \times 41$
8	$2^3$	33	$3 \times 11$	58	$2 \times 29$	83	
9	$3^2$	34	$2 \times 17$	59		84	$2^2 \times 3 \times 7$
10	$2 \times 5$	35	$5 \times 7$	60	$2^2 \times 3 \times 5$	85	$5 \times 17$
11		36	$2^2 \times 3^2$	61		86	$2 \times 43$
12	$2^2 \times 3$	37		62	$2 \times 31$	87	$3 \times 29$
13		38	$2 \times 19$	63	$3^2 \times 7$	88	$2^{3} \times 11$
14	$2 \times 7$	39	$3 \times 13$	64	$2^6$	89	
15	$3 \times 5$	40	$2^3 \times 5$	65	$5 \times 13$	90	$2 \times 3^2 \times 5$
16	24	41		66	$2 \times 3 \times 11$	91	$7 \times 13$
17		42	$2 \times 3 \times 7$	67		92	$2^2 \times 23$
18	$2 \times 3^2$	43		68	$2^2 \times 17$	93	$3 \times 31$
19		44	$2^2 \times 11$	69	$3 \times 23$	94	$2 \times 47$
20	$2^2 \times 5$	45	$3^2 \times 5$	70	$2 \times 5 \times 7$	95	$5 \times 19$
21	$3 \times 7$	46	$2 \times 23$	71		96	$2^5 \times 3$
22	$2 \times 11$	47		72	$2^3 \times 3^2$	97	
23		48	$2^4 \times 3$	73		98	$2 \times 7^2$
24	$2^3 \times 3$	49	$7^2$	74	$2 \times 37$	99	$3^2 \times 11$
25	$5^2$	50	$2 \times 5^2$	75	$3 \times 5^2$	100	$2^2 \times 5^2$

401		101		1	- 00	101	
101		131		161		191	
102	$2 \times 3 \times 17$		$2^2 \times 3 \times 11$			192	$2^6 \times 3$
103		133	$7 \times 19$	163		193	
104	$2^{\circ} \times 13$	134	$2 \times 67$	164	$2^2 \times 41$	194	$2 \times 97$
105	$3 \times 5 \times 7$	135	$3^3 \times 5$	165	$3 \times 5 \times 11$	195	$3 \times 5 \times 13$
106	$2 \times 53$	136	$2^3 \times 17$	166	$2 \times 83$	196	$2^2 \times 7^2$
107		137		167		197	
108	$2^2 \times 3^3$	138	$2 \times 3 \times 23$	168	$2^{\circ} \times 3 \times 7$	198	$2 \times 3^2 \times 11$
109		139		169	$13^{2}$	199	
110	$2 \times 5 \times 11$	140	$2^2 \times 5 \times 7$	170	$2 \times 5 \times 17$	200	$2^3 \times 5^2$
111	$3 \times 37$	141	$3 \times 47$	171	$3^2 \times 19$	201	$3 \times 67$
112	$2^4 \times 7$	142	$2 \times 71$	172	$2^2 \times 43$	202	$2 \times 101$
113		143	$11 \times 13$	173		203	$7 \times 29$
114	$2 \times 3 \times 19$	144	$2^4 \times 3^2$	174	$2 \times 3 \times 29$	204	$2^2 \times 3 \times 17$
115	$5 \times 23$	145	$5 \times 29$	175	$5^2 \times 7$	205	5×41
116	$2^2 \times 29$	146	$2 \times 73$	176	$2^4 \times 11$	206	$2 \times 103$
117	$3^2 \times 13$	147	$3 \times 7^2$	177	$3 \times 59$	207	$3^{2} \times 23$
118	$2 \times 59$	148	$2^2 \times 37$	178	$2 \times 89$	208	$2^4 \times 13$
119	$7 \times 17$	149		179		209	11×19
120	$2^3 \times 3 \times 5$	150	$2 \times 3 \times 5^2$	180	$2^2 \times 3^2 \times 5$	210	$2 \times 3 \times 5 \times 7$
121	$11^{2}$	151		181		211	
122	$2 \times 61$	152	$2^{3} \times 19$	182	$2 \times 7 \times 13$	212	$2^2 \times 53$
123	$3 \times 41$	153	$3^2 \times 17$	183	$3 \times 61$	213	$3 \times 71$
124	$2^2 \times 31$	154	$2 \times 7 \times 11$	184	$2^3 \times 23$	214	$2 \times 107$
125	$5^3$	155	$5 \times 31$	185	$5 \times 37$	215	$5 \times 43$
126	$2 \times 3^2 \times 7$	156	$2^2 \times 3 \times 13$	186	$2 \times 3 \times 31$	216	$2^3 \times 3^3$
127		157		187	11×17	217	$7 \times 31$
128	27	158	$2 \times 79$	188	$2^2 \times 47$	218	$2 \times 109$
129	$3 \times 43$	159	$3 \times 53$	189	$3^3 \times 7$	219	$3 \times 73$
130	$2 \times 5 \times 13$	160	$2^5 \times 5$	190	$2 \times 5 \times 19$	220	$2^2 \times 5 \times 11$

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221	$13 \times 17$	251		281		311	
222	$2 \times 3 \times 37$	252	$2^2 \times 3^2 \times 7$	282	$2 \times 3 \times 47$	312	$2^3 \times 3 \times 13$
223		253	$11 \times 23$	283		313	
224	$2^5 \times 7$	254	$2 \times 127$	284	$2^2 \times 71$	314	$2 \times 157$
225	$3^2 \times 5^2$	255	$3 \times 5 \times 17$	285	$3 \times 5 \times 19$	315	$3^2 \times 5 \times 7$
226	$2 \times 113$	256	$2^8$	286	$2 \times 11 \times 13$	316	$2^2 \times 79$
227		257		287	$7 \times 41$	317	
228	$2^2 \times 3 \times 19$	258	$2 \times 3 \times 43$	288	$2^5 \times 3^2$	318	$2 \times 3 \times 53$
229		259	$7 \times 37$	289	$17^2$	319	$11 \times 29$
230	$2 \times 5 \times 23$	260	$2^2 \times 5 \times 13$	290	$2 \times 5 \times 29$	320	$2^6 \times 5$
231	$3 \times 7 \times 11$	261	$3^2 \times 29$	291	$3 \times 97$	321	$3 \times 107$
232	$2^3 \times 29$	262	$2 \times 131$	292	$2^2 \times 73$	322	$2 \times 7 \times 23$
233		263		293		323	$17 \times 19$
234	$2 \times 3^2 \times 13$	264	$2^3 \times 3 \times 11$	294	$2 \times 3 \times 7^2$	324	$2^2 \times 3^4$
235	$5 \times 47$	265	$5 \times 53$	295	$5 \times 59$	325	$5^2 \times 13$
236	$2^2 \times 59$	266	$2 \times 7 \times 19$	296	$2^3 \times 37$	326	$2 \times 163$
237	$3 \times 79$	267	$3 \times 89$	297	$3^3 \times 11$	327	$3 \times 109$
238	$2 \times 7 \times 17$	268	$2^2 \times 67$	298	$2 \times 149$	328	$2^3 \times 41$
239		269		299	$13 \times 23$	329	$7 \times 47$
240	$-2^4 \times 3 \times 5$	270	$2 \times 3^3 \times 5$	300	$2^2 \times 3 \times 5^2$	330	$2\times3\times5\times11$
241		271		301	$7 \times 43$	331	,
242	$2 \times 11^2$	272	$2^4 \times 17$	302	$2 \times 151$	332	$2^2 \times 83$
243	$\mathfrak{R}^5$	273	$3 \times 7 \times 13$	303	$3 \times 101$	333	$3^2 \times 37$
244	$2^2 \times 61$	274	$2 \times 137$	304	$2^4 \times 19$	334	$2 \times 167$
245	$5 \times 7^2$	275	$5^2 \times 11$	305	$5 \times 61$	335	$5 \times 67$
246	$2 \times 3 \times 41$	276	$2^2 \times 3 \times 23$	306	$2 \times 3^2 \times 17$	336	$2^4 \times 3 \times 7$
247	$13 \times 19$	277		307		337	
248	$2^3 \times 31$	278	$2 \times 139$	308	$2^2 \times 7 \times 11$	338	$2 \times 13^2$
249	$3 \times 83$	279	$3^2 \times 31$	309	$3 \times 103$	339	$3 \times 113$
250	$2 \times 5^3$	280	$2^3 \times 5 \times 7$	310	$2 \times 5 \times 31$	340	$2^2 \times 5 \times 17$

341	11×31	371	$7 \times 53$	401		431	
342	$2 \times 3^2 \times 19$	372	$2^2 \times 3 \times 31$	402	$2 \times 3 \times 67$	432	$2^4 \times 3^3$
343	73	373		403	$13 \times 31$	433	
344	$2^3 \times 43$	374	$2 \times 11 \times 17$	404	$2^2 \times 101$	434	$2 \times 7 \times 31$
345	$2 \times 5 \times 23$	375	$3 \times 5^{3}$	405	$3^4 \times 5$	435	$3 \times 5 \times 29$
346	$2 \times 173$	376	$2^3 \times 47$	406	$2 \times 7 \times 29$	436	$2^2 \times 109$
347		377	13×19	407	$11 \times 37$	437	$19 \times 23$
348	$2^2 \times 3 \times 29$	378	$2 \times 3^3 \times 7$	408	$2^{\circ} \times 3 \times 17$	438	$2 \times 3 \times 73$
349		379		409		439	
350	$2 \times 5^2 \times 7$	380	$2^2 \times 5 \times 19$	410	$2 \times 5 \times 41$	440	$2^3 \times 5 \times 11$
351	$3^3 \times 13$	381	$3 \times 127$	411	$3 \times 137$	441	$3^2 \times 7^2$
352	$2^5 \times 11$	382	$2 \times 191$	412	$2^2 \times 103$	412	$2 \times 13 \times 17$
353		383		413	$7 \times 59$	443	
354	$2 \times 3 \times 59$	384	$2^7 \times 3$	414	$2 \times 3^2 \times 23$	444	$2^2 \times 3 \times 37$
355	$5 \times 71$	385	$5 \times 7 \times 11$	415	$5 \times 83$	445	$5 \times 89$
356	$2^2 \times 89$	386	$2 \times 193$	416	$2^5 \times 13$	446	$2 \times 223$
357	$3 \times 7 \times 17$	387	$3^2 \times 43$	417	$3 \times 139$	447	$3 \times 149$
358	$2 \times 179$	388	$2^2 \times 97$	418	$2 \times 11 \times 19$	448	$2^6 \times 7$
359		389		419	~	449	
360	$2^3 \times 3^2 \times 5$	390	$2\times3\times5\times13$	420	$2^2 \times 3 \times 5 \times 7$	450	$2 \times 3^2 \times 5^2$
361	$19^{2}$	391	$17 \times 23$	421		451	11×41
362	$2 \times 181$	392	$2^3 \times 7^2$	422	2×211	452	$2^2 \times 113$
363	$3 \times 11^2$	393	$3 \times 131$	423	$3^2 \times 47$	453	$3 \times 151$
364	$2^2 \times 7 \times 13$	394	$2 \times 197$	424	$2^3 \times 53$	454	$2 \times 227$
365	$5 \times 73$	395	$5 \times 79$	425	$5^2 \times 17$	455	$5 \times 7 \times 13$
366	$2 \times 3 \times 61$	396	$2^2 \times 3^2 \times 11$	426	$2 \times 3 \times 71$	456	$2^3 \times 3 \times 19$
367		397		427	$7 \times 61$	457	
368	$2^4 \times 23$	398	$2 \times 199$	428	$2^2 \times 107$	458	$2 \times 229$
369	$3^2 \times 41$	399	$3 \times 7 \times 19$	429	$3 \times 11 \times 13$	459	$3^3 \times 17$
370	$2 \times 5 \times 37$	400	$2^4 \times 5^2$	430	$2 \times 5 \times 43$	460	$2^2 \times 5 \times 23$

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461		491		521		551	19×29
462	$2\times3\times7\times11$	492	$2^2 \times 3 \times 41$	522	$2 \times 3^2 \times 29$	552	$2^3 \times 3 \times 23$
463		493	$17 \times 29$	523		553	7×79
464	$2^4 \times 29$	494	$2 \times 13 \times 19$	524	$2^2 \times 131$	554	$2 \times 277$
465	$3 \times 5 \times 31$	495	$3^2 \times 5 \times 11$	525	$3 \times 5^2 \times 7$	555	$3 \times 5 \times 37$
466	$2 \times 233$	496	$2^4 \times 31$	526	$2 \times 263$	556	$2^2 \times 139$
467	,	497	$7 \times 71$	527	$17 \times 31$	557	
468	$2^2 \times 3^2 \times 13$	498	$2 \times 3 \times 83$	528	$2^4 \times 3 \times 11$	558	$2 \times 3^2 \times 31$
469	$7 \times 67$	499		529	$23^{2}$	559	$13 \times 43$
470	$2 \times 5 \times 47$	500	$2^2 \times 5^3$	530	$2 \times 5 \times 53$	560	$2^4 \times 5 \times 7$
471	$3 \times 157$	501	$3 \times 167$	531	$3^2 \times 59$	561	$3 \times 11 \times 17$
472	$2^3 \times 59$	502	$2 \times 251$	532	$2^2 \times 7 \times 19$	562	$2 \times 281$
473	11×43	503		533	13×41	563	
474	$2 \times 3 \times 79$	504	$2^3 \times 3^2 \times 7$	534	$2 \times 3 \times 89$	564	$2^2 \times 3 \times 47$
475	$5^2 \times 19$	505	$5 \times 101$	535	$5 \times 107$	565	$5 \times 113$
476	$2^2 \times 7 \times 17$	506	$2 \times 11 \times 23$	536	$2^3 \times 67$	566	$2 \times 283$
477	$3^2 \times 53$	507	$3 \times 13^2$	537	$3 \times 179$	567	$3^4 \times 7$
478	$2 \times 239$	508	$2^2 \times 127$	538	$2 \times 269$	568	$2^3 \times 71$
479		509		539	$7^2 \times 11$	569	
480	$2^5 \times 3 \times 5$	510	$2\times3\times5\times17$	540	$2^2 \times 3^3 \times 5$	570	$2\times3\times5\times19$
481	$13 \times 37$	511	$7 \times 73$	541		571	
482	$2 \times 241$	512	$2^9$	542	$2 \times 271$	572	$2^2 \times 11 \times 13$
483	$3 \times 7 \times 23$	513	$3^3 \times 19$	543	$3 \times 181$	573	$3 \times 191$
484	$2^2 \times 11^2$	514	$2 \times 257$	544	$2^5 \times 17$	574	$2 \times 7 \times 41$
485	$5 \times 97$	515	$5 \times 103$	545	5×109	575	$5^2 \times 23$
486	$2 \times 3^5$	516	$2^2 \times 3 \times 43$	546	$2\times3\times7\times13$	576	$2^6 \times 3^2$
487		517	$11 \times 47$	547		577	
488	$2^3 \times 61$	518	$2 \times 7 \times 37$	548	$2^2 \times 137$	578	$2 \times 17^2$
489	$3 \times 163$	519	$3 \times 173$	549	$3^2 \times 61$	579	$3 \times 193$
490	$2 \times 5 \times 7^2$	520	$2^3 \times 5 \times 13$	550	$2 \times 5^2 \times 11$	580	$2^2 \times 5 \times 29$

$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	37   5 <sup>2</sup>   13 <sup>2</sup>   113   77   × 17   27
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$\begin{bmatrix} 37 \\ 5^2 \\ 13^2 \\ 113 \\ 07 \\ \times 17 \\ 27 \end{bmatrix}$
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$5^{2}$ $13^{2}$ $113$ $113$ $113$ $113$ $113$ $113$ $113$ $113$
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$5^{2}$ $13^{2}$ $113$ $113$ $113$ $113$ $113$ $113$ $113$ $113$ $113$
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$13^{2}$ $113$ $07$ $\times 17$ $27$
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	×17 27
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	)7 ×17 27
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	)7 ×17 27
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	×17
$\begin{bmatrix} 591 & 3 \times 197 & 621 & 3^3 \times 23 & 651 & 3 \times 7 \times 31 & 681 & 3 \times 2 \end{bmatrix}$	27
	1
$592$ $2^4 \times 37$ $622$ $2 \times 311$ $652$ $2^2 \times 163$ $682$ $2 \times 11$	×31
$623  7 \times 89  653$	
$594  2 \times 3^{3} \times 11  624  2^{4} \times 3 \times 13  654  2 \times 3 \times 109  684  2^{2} \times 3^{2}$	×19
$595  5 \times 7 \times 17  625  5^4  655  5 \times 131  685  5 \times 1$	37
$596   2^2 \times 149   626   2 \times 313   656   2^4 \times 41   686   2 \times$	73
$597$ $3 \times 199$ $627$ $3 \times 11 \times 19$ $657$ $3^2 \times 73$ $687$ $3 \times 2$	29
$598 \ 2 \times 13 \times 23 \ 628 \ 2^2 \times 157 \ 658 \ 2 \times 7 \times 47 \ 688 \ 2^4 \times 500 \ 2$	13
<b>599</b> 629 $17 \times 37$ <b>659 689</b>	
600 $2^3 \times 3 \times 5^2$ 630 $2 \times 3^2 \times 5 \times 7$ 660 $2^2 \times 3 \times 5 \times 11$ 690 $2 \times 3 \times 5$	×23
601 631 661 691	
$602 \ 2 \times 7 \times 43 \ 632 \ 2^{\circ} \times 79 \ 662 \ 2 \times 331 \ 692 \ 2^{\circ} \times 10^{\circ}$	73
603 $3^2 \times 67$ 633 $3 \times 211$ 663 $3 \times 13 \times 17$ 693 $3^2 \times 7$	×11
604 $2^2 \times 151$ 634 $2 \times 317$ 664 $2^3 \times 83$ 694 $2 \times 3$	47
605 $5 \times 11^2$ 635 $5 \times 127$ 665 $5 \times 7 \times 19$ 695 $5 \times 1$	39
606 $2 \times 3 \times 101$ 636 $2^2 \times 3 \times 53$ 666 $2 \times 3^2 \times 37$ 696 $2^3 \times 3$	×29
<b>607</b> 637 $7^2 \times 13$ 667 $23 \times 29$ 697 $17 \times$	41
608 $2^5 \times 19$ 638 $2 \times 11 \times 29$ 668 $2^2 \times 167$ 698 $2 \times 3$	49
609 $3 \times 7 \times 29$ 639 $3^2 \times 71$ 669 $3 \times 223$ 699 $3 \times 2$	33
610 $2 \times 5 \times 61$ 640 $2^7 \times 5$ 670 $2 \times 5 \times 67$ 700 $2^2 \times 5^2$	× 7

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701		731	$17 \times 43$	761		791	7×113
702	$2 \times 3^3 \times 13$	732	$2^2 \times 3 \times 61$	762	$2 \times 3 \times 127$	792	$2^3 \times 3^2 \times 11$
703	$19 \times 37$	733		763	$7 \times 109$	793	$13 \times 61$
704	$2^{6} \times 11$	734	$2 \times 367$	764	$2^2 \times 191$	794	$2 \times 397$
705	$3 \times 5 \times 47$	735	$3 \times 5 \times 7^2$	765	$3^2 \times 5 \times 17$	795	$3 \times 5 \times 53$
706	$2 \times 353$	736	$2^5 \times 23$	766	$2 \times 383$	796	$2^2 \times 199$
707	$7 \times 101$	737	$11 \times 67$	767	$13 \times 59$	797	
708	$2^2 \times 3 \times 59$	738	$2 \times 3^2 \times 41$	768	$2^8 \times 3$	798	$2\times3\times7\times19$
709		739		769		799	$17 \times 47$
710	$2 \times 5 \times 71$	740	$2^2 \times 5 \times 37$	770	$2 \times 5 \times 7 \times 11$	800	$2^5 \times 5^2$
711	$3^2 \times 79$	741	$3 \times 13 \times 19$	771	$3 \times 257$	801	$3^2 \times 89$
712	$2^3 \times 89$	742	$2 \times 7 \times 53$	772	$2^2 \times 193$	802	$2 \times 401$
713	$23 \times 31$	743		773		803	$11 \times 73$
714	$2\times3\times7\times17$	744	$2^3 \times 3 \times 31$	774	$2 \times 3^2 \times 43$	804	$2^2 \times 3 \times 67$
715	$5 \times 11 \times 13$	745	$5 \times 149$	775	$5^2 \times 31$	805	$5 \times 7 \times 23$
716	$2^2 \times 179$	746	$2 \times 373$	776	$2^3 \times 97$	806	$2 \times 13 \times 31$
717	$3 \times 239$	747	$3^2 \times 83$	777	$3 \times 7 \times 37$	807	$3 \times 269$
718	$2 \times 359$	748	$2^2 \times 11 \times 17$	778	$2 \times 389$	808	$2^{3} \times 101$
719		749	$7 \times 107$	779	$19 \times 41$	809	
720	$2^4 \times 3^2 \times 5$	750	$2 \times 3 \times 5^3$	780	$2^2 \times 3 \times 5 \times 13$	810	$2 \times 3^4 \times 5$
721	$7 \times 103$	751		781	11×71	811	
722	$2 \times 19^2$	752	$2^4 \times 47$	782	$2 \times 17 \times 23$	812	$2^2 \times 7 \times 29$
723	$3 \times 241$	753	$3 \times 251$	783	$3^3 \times 29$	813	$3 \times 271$
724	$2^2 \times 181$	754	$2 \times 13 \times 29$	784	$2^4 \times 7^2$	814	$2 \times 11 \times 37$
725	$5^2 \times 29$	755	$5 \times 151$	785	$5 \times 157$	815	$5 \times 163$
726	$2 \times 3 \times 11^2$	756	$2^2 \times 3^3 \times 7$	786	$2 \times 3 \times 131$	816	$2^4 \times 3 \times 17$
727		757		787		817	$19 \times 43$
728	$2^3 \times 7 \times 13$	758	$2 \times 379$	788	$2^2 \times 197$	818	$2 \times 409$
729	$3^6$	759	$3 \times 11 \times 23$	789	$3 \times 263$	819	$3^2 \times 7 \times 13$
730	$2 \times 5 \times 73$	760	$2^3 \times 5 \times 19$	790	$2 \times 5 \times 79$	820	$2^2 \times 5 \times 41$
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821		851	$23 \times 37$	881	1	911	
822	$2 \times 3 \times 137$	852	$2^2 \times 3 \times 71$	882	$2\times3^2\times7^2$	912	$2^4 \times 3 \times 19$
823		853		883		913	11×83
824	$2^3 \times 103$	854	$2 \times 7 \times 61$	884	$2^2 \times 13 \times 17$	914	$2 \times 457$
825	$3 \times 5^2 \times 11$	855	$3^2 \times 5 \times 19$	885	$3 \times 5 \times 59$	915	$3 \times 5 \times 61$
826	$2 \times 7 \times 59$	856	$2^3 \times 107$	886	$2 \times 443$	916	$2^2 \times 229$
827		857		887		917	$7 \times 131$
828	$2^2 \times 3^2 \times 23$	858	$2 \times 3 \times 11 \times 13$	888	$2^3 \times 3 \times 37$	918	$2 \times 3^3 \times 17$
829		859		889	$7 \times 127$	919	
830	$2 \times 5 \times 83$	860	$2^2 \times 5 \times 43$	890	$2 \times 5 \times 89$	920	$2^3 \times 5 \times 23$
831	$3 \times 277$	861	$3 \times 7 \times 41$	891	$3^4 \times 11$	921	$3 \times 307$
832	$2^6 \times 13$	862	$2 \times 431$	892	$2^2 \times 223$	922	$2 \times 461$
833	$7^2 \times 17$	863		893	$19 \times 47$	923	$13 \times 71$
834	$2 \times 3 \times 139$	864	$2^5 \times 3^3$	894	$2 \times 3 \times 149$	924	$2^2 \times 3 \times 7 \times 11$
835	$5 \times 167$	865	$5 \times 173$	895	$5 \times 179$	925	$5^2 \times 37$
836	$2^2 \times 11 \times 19$	866	$2 \times 433$	896	$2^7 \times 7$	926	$2 \times 463$
837	$3^3 \times 31$	867	$3 \times 17^2$	897	$3 \times 13 \times 23$	927	$3^2 \times 103$
838	$2 \times 419$	868	$2^2 \times 7 \times 31$	898	$2 \times 449$	928	$2^5 \times 29$
839		869	$11 \times 79$	899	$29 \times 31$	929	
840	$2^3 \times 3 \times 5 \times 7$	870	$2\times3\times5\times29$	900	$2^2 \times 3^2 \times 5^2$	930	$2\times3\times5\times31$
841	$29^{2}$	871	$13 \times 67$	901	$17 \times 53$	931	$7^2 \times 19$
842	$2 \times 421$	872	$2^3 \times 109$	902	$2 \times 11 \times 41$	932	$2^2 \times 233$
843	$3 \times 281$	873	$3^2 \times 97$	903	$3 \times 7 \times 43$	933	$3 \times 311$
844	$2^2 \times 211$	874	$2 \times 19 \times 23$	904	$2^{3} \times 113$	934	$2 \times 467$
845	$5 \times 13^2$	875	$5^3 \times 7$	905	$5 \times 181$	935	$5 \times 11 \times 17$
846	$2 \times 3^2 \times 47$	876	$2^2 \times 3 \times 73$	906	$2 \times 3 \times 151$	936	$2^3 \times 3^2 \times 13$
847	$7 \times 11^2$	877		907		937	
848	$2^4 \times 53$	878	$2 \times 439$	908	$2^2 \times 227$	938	$2 \times 7 \times 67$
849	$3 \times 283$	879	$3 \times 293$	909	$3^2 \times 101$	939	$3 \times 313$
850	$2 \times 5^2 \times 17$	880	$2^4 \times 5 \times 11$	910	$2\times5\times7\times13$	940	$2^2 \times 5 \times 47$

				11			
941		956	$2^2 \times 239$	971		986	$2 \times 17 \times 29$
942	$2 \times 3 \times 157$	957	$3 \times 11 \times 29$	972	$2^2 \times 3^5$	987	$3 \times 7 \times 47$
943	$23 \times 41$	958	$2 \times 479$	973	$7 \times 139$	988	$2^2 \times 13 \times 19$
944	$2^4 \times 59$	959	$7 \times 137$	974	$2 \times 487$	989	$23 \times 43$
945	$3^3 \times 5 \times 7$	960	$2^6 \times 3 \times 5$	975	$3 \times 5^2 \times 13$	990	$2\times3^2\times5\times11$
946	$2 \times 11 \times 43$	961	$31^{2}$	976	$2^4 \times 61$	991	
947		962	$2 \times 13 \times 37$	977		992	$2^5 \times 31$
948	$2^2 \times 3 \times 79$	963	$3^2 \times 107$	978	$2 \times 3 \times 163$	993	$3 \times 331$
949	$13 \times 73$	964	$2^2 \times 241$	979	$11 \times 89$	994	$2 \times 7 \times 71$
950	$2 \times 5^2 \times 19$	965	$5 \times 193$	980	$2^2 \times 5 \times 7^2$	995	$5 \times 199$
951	$3 \times 317$	966	$2\times3\times7\times23$	981	$3^2 \times 109$	996	$2^2 \times 3 \times 83$
952	$2^3 \times 7 \times 17$	967	,	982	$2 \times 491$	997	
953		968	$2^{\circ} \times 11^{2}$	983		998	$2 \times 499$
954	$2 \times 3^2 \times 53$	969	$3 \times 17 \times 19$	984	$2^3 \times 3 \times 41$	999	$3^3 \times 37$
955	$5 \times 191$	970	$2 \times 5 \times 97$	985	$5 \times 197$	1000	$2^3 \times 5^3$

#### CHAPTER VI.

#### INTERNAL GEARING.

#### PART A.-INTERNAL SPUR GEARING.

(Figs. 12, 13, 14, 15, 16.)

A little consideration will show that a tooth of an internal or annular gear is the same as the space of a spur—external gear.

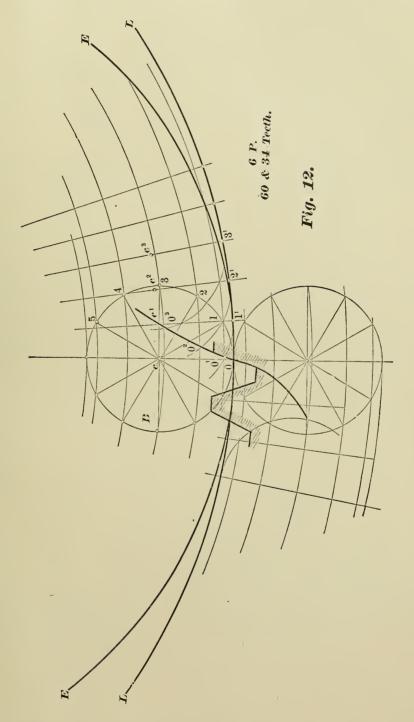
We prefer the epicycloidal form of tooth in this class of gearing to the involute form, for the reason that the difficulties in overcoming the interference of gear teeth in the involute system are considerable. Special constructions are required when the difference between the number of teeth in gear and pinion is small.

In using the system of epicycloidal form of tooth in which the gear of 15 teeth has radial flanks, this difference must be at least 15 teeth, if the teeth have both faces and flanks. Gears fulfilling this condition present no difficulties. Their pitch diameters are found as in regular spur gears, and the inside diameter is equal to the pitch diameter, less twice the addendum.

If, however, this difference is less than 15, say 6, or 2, or 1, then we may construct the tooth outline (based on the epicycloidal system) in two different ways.

First Method.—To explain this method better, let us suppose the case as in Fig. 12, in which the difference between gear and pinion is more than 15 teeth. Here the point o of the describing circle B (the diameter of which in the best practice of the present day is equal to the pitch radius of a 15 tooth gear, of the same pitch as the gears in question) generates the cycloid o, o¹, o², o³, etc., when rolling on pitch circle L L of gear, forming the face of tooth; and when rolling on the outside of L L the flank of the tooth. In like manner is the face and flank of the pinion tooth produced by B rolling outside and inside of E E (pitch circle of pinion). A little study

of Fig. 12 (in which the face and flank of a gear tooth are produced) will show the describing circle B divided into 12



equal parts and circles laid through these points (1, 2, 3, etc.), concentric with L L. We now lay off on L L the distances 0-1, 1-2, 2-3, etc., of the circumference of B, and obtain points

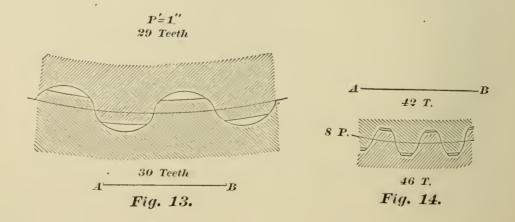
 $I^1$ ,  $I^2$ ,  $I^3$ , etc. [Ordinarily it is sufficient to use the chord.] It will now readily be seen that B in rolling on L L will successively come in contact with  $I^1$ ,  $I^2$ ,  $I^3$ , etc.,  $I^2$  meanwhile moving to  $I^2$ ,  $I^2$ ,  $I^3$ , etc. (points on radii through  $I^2$ ,  $I^3$ , etc.), and the generating point o advancing to  $I^2$ ,  $I^2$ ,  $I^3$ , etc., being the intersections of B with  $I^2$ ,  $I^2$ ,  $I^3$ , etc., as centers and the circles laid through  $I^2$ ,  $I^2$ ,  $I^3$ , etc. Points  $I^2$ ,  $I^3$ , etc., connected with a curve give the face of the tooth; in like manner the flank is obtained.

In this manner the form of tooth is obtained, when the difference of teeth in gear and pinion is less than 15, with the exception that the diameter of describing circle B

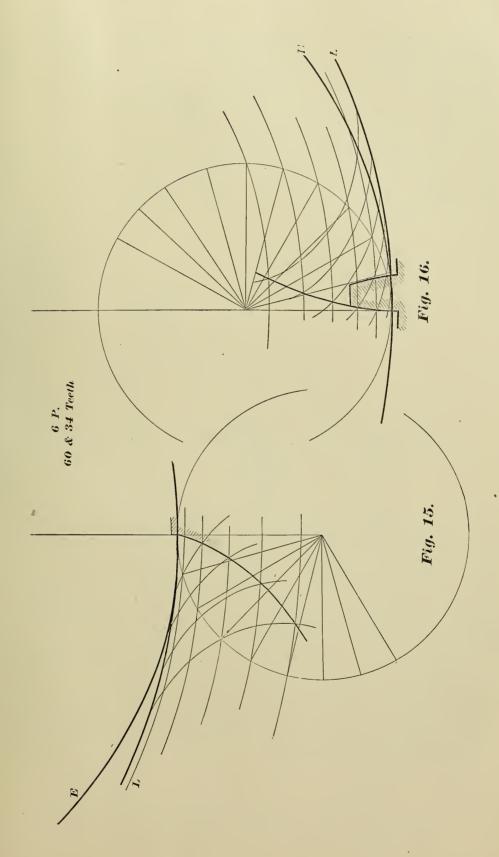
$$= \frac{1}{2} \left( \frac{N - n}{P} \right)$$

where P = diametral pitch, N and n number of teeth in gears.

The distances of the tooth above and below the pitch line as well as the thickness t are determined as in regular spur gears by the pitch, except when the difference in gear and pinion is very small, where we obtain a short tooth, as in Figs. 13 and 14. In such a case the height of tooth is arbitrary and only conditioned by the curve. In internal gears it is best to allow more clearance at bottom of tooth than in ordinary spur gears.



In a construction of this kind it is suggested to draw the tooth outline many times full size and reduce by photography. An equally multiplied line A B will help in reducing.



Second Method.—The difference between gear and pinion being very small, it is sometimes desirable to obtain a smooth action by avoiding what is termed the "friction of approaching action."\* This is done, the pinion driving, by giving gear only flanks, Fig. 15, and the gear driving, by giving gear only faces, Fig. 16. In both these cases we have but one describing circle, whose diameter is equal to the difference of the two pitch diameters. The construction of the curve is precisely the same as described under A. The describing circle has been divided into 24 parts simply for the sake of greater accuracy.

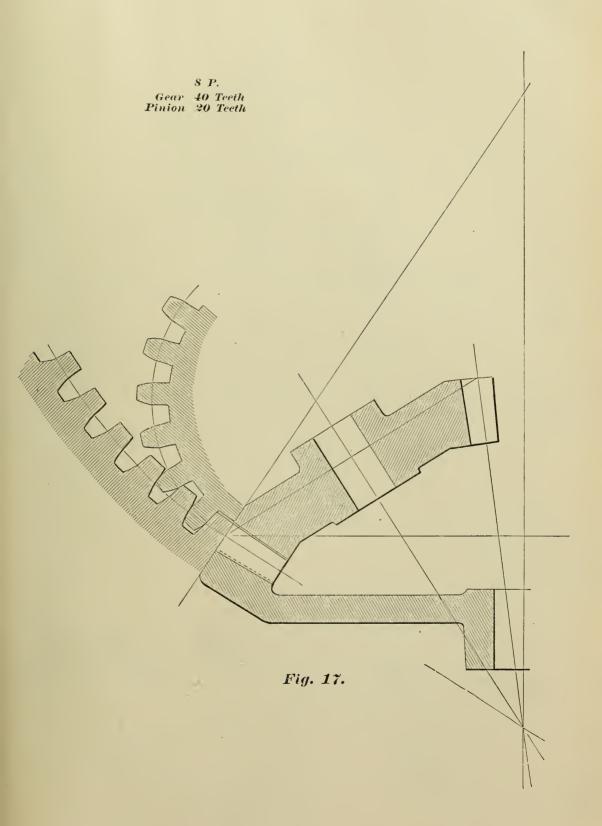
### PART B.-INTERNAL BEVEL GEARS.

(Fig. 17.)

The pitch surfaces of bevel gears are cones whose apexes are at a common point, rolling upon each other. The tooth forms for any given pair of bevel gears are the same as for a pair of spur gears (of same pitch) whose pitch radii are equal to the respective apex distances of the normal cones (i. e., cones whose elements are perpendicular upon the elements of the bevel gear pitch cones). (Compare Fig 19, page 68.)

The same is true of internal bevel gears, with the modification that here one of the pitch cones rolls inside of the other. The spur gears to whose tooth forms the forms of the bevel gear teeth correspond, resolve themselves into internal spur gears (Fig. 17). The problem is now to be solved as indicated in the first part of this chapter.

<sup>#</sup> McCord, Kinematics, pages 107, 108.



### CHAPTER VII.

### GEAR PATTERNS.

(Fig. 18.)

To place in bevel gears the best iron where it belongs, the tooth side of the pattern should always be in the nowel, no matter of what shape the hubs are.

Hubs, if short, may be left solid on web; if long they should be made loose. A long hub should go on a tapering arbor, to prevent tipping in the sand. 1° taper for draft on hubs when loose, and 3° when solid is considered sufficient.

Coreprints as a rule are made separate, partly to allow the pattern to be turned on an arbor, partly for convenience, should it be desirable to use different sizes.

Put rap- and draw-holes as near to center as possible. Referring to Fig. 18, make L = D for D from  $\frac{3}{4}$ " to  $\frac{1}{2}$ ", or even more, should hubs be very long. Otherwise if D is more than  $\frac{1}{2}$ " leave  $L = \frac{1}{2}$ ".

Iron pattern before using should be marked, rusted and waxed.

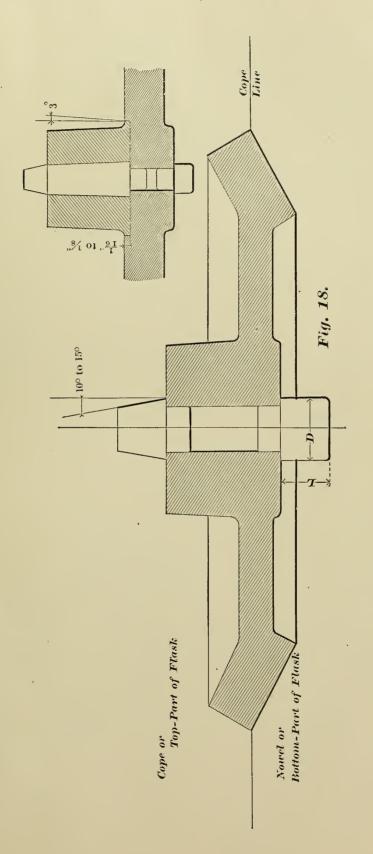
Shrinkage—For cast-iron, 
$$\frac{1}{8}$$
" per foot.  
For brass,  $\frac{3}{18}$ " "

Cast-iron gears, especially arm gears, do not always shrink 1/8" per foot. In making iron patterns the following allowances have been found useful:

Up to 12" diameter allow no shrink.

From 12" to 18" " " 1/3 regular shrink.
" 18" to 24" " " 1/2 " "
" 24" to 48" " " 2/3 " "
Above 48" " " .10" " "

for cast-iron.



If in gears the teeth are to be cast, the tooth thickness t in the pattern is made smaller than called for by the pitch, to avoid binding of the teeth when cast. No definite rule can be given, as the practice varies on this point. For the different diametral pitches we would advise making t smaller by an amount expressed in inches, as given in the following table:

DIAM. PITCH.	Amount t is Smaller.	DIAM, РІТСН.	AMOUNT t IS SMALLER.		
16	.010′′	5	.020"		
12	.012"	4	.022''		
10	.014"	3			
8	.016"	2	.030"		
6	.018"	I	.040′′		

### CHAPTER VIII.

## DIMENSIONS AND FORM FOR BEVEL GEAR CUTTERS.

(Fig. 19.)

The data needed to determine the form and thickness of a bevel gear cutter are the following:

P = pitch.

N = number of teeth in large gear.

n = number of teeth in small gear.

F = length of face of tooth, measured on pitch line.

After having laid out a diagram of the pitch cones a b c and a b f, and laid off the width of face, the problem resolves itself into two parts:

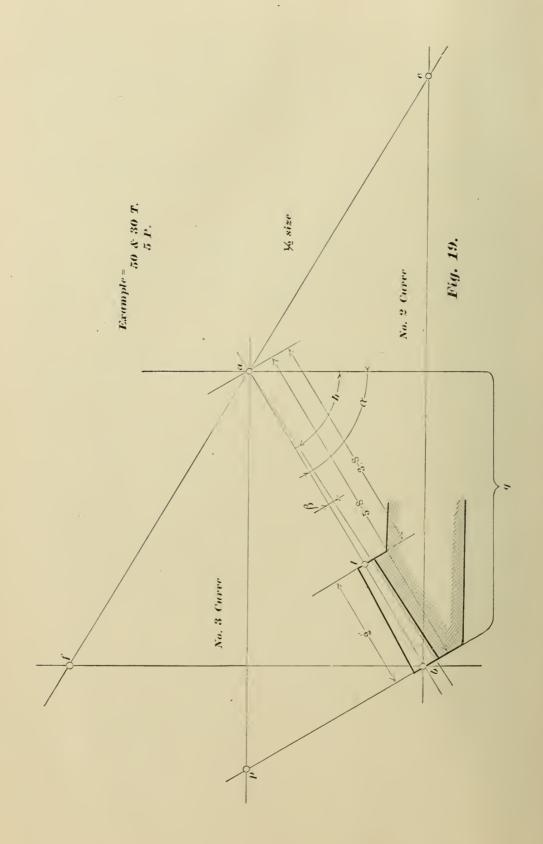
PART I.—DETERMINE PROPER CURVE FOR CUTTER.

It will be remembered that in the involute system of cutters (the only one used for bevel gears that are cut with rotary cutter), a set of eight different cutters is made for each pitch, numbering from No. 1 to No. 8, and cutting from a rack to 12 teeth. Each number represents the form of a cutter suitable to cut the indicated number of teeth. For instance, No. 4 cutter (No. 4 curve) will cut 26 to 34 teeth. In order to find the curve to be used for gear and pinion we simply construct the normal pitch cones by erecting the perpendicular p q through p, Fig. 19. We now measure the lines p p and p p, and taking them as radii, multiplying each by 2 and P we obtain a number of teeth for which cutters of proper curves may be selected. From example we have:

Gear: 
$$b \ q = 9\frac{3}{4}$$
";  $2 \times P \times 9.75 = 97$  T No. 2 curve.   
Pinion:  $b \ p = 3\frac{1}{2}$ ";  $2 \times P \times 3.5 = 35$  T No. 3 curve.

The eight cutters which are made in the involute system for each pitch are as follows:

No.	Ι	will cut	wheels	from	135	teeth	to	a ra	.ck.
"	2	66		66	55	66	66	134	teeth.
"	3	"	. 6	"	35	"	"	54	"
66	4	"	"	"	26	"	"	34	66
"		"	"	"		"	"	25	66
"	6	44	"	66		66	"	20	"
"	7	"	"	46		"	"	16	66
66	8	"	"	"	12	"	"	13	44



### PART II. - DETERMINE THICKNESS OF CUTTER.

It is very evident that a bevel gear cutter cannot be thicker than the width of the space at small end of tooth; the practice is to make cutter .oo5" thinner. Theoretically the cutting angle (h) is equal to pitch angle less angle of bottom (or  $h = \alpha - \beta'$ ). Practically, however, better results are obtained by making  $h = \alpha - \beta$  (substituting angle of top for angle of bottom), and in calculating the depth at small end, to add the full clearance (f) to the obtained working depth, giving equal amount of clearance at large and small end. This is done to obtain a tooth thinner at the top and more curved. As the small end of tooth determines the thickness of cutter, we shall have to find the tooth part values at small end. From the diagram it will be seen that the values at large end are to those at small end as their respective apex distances (a b and a l). numerical values of these can be taken from the diagram and the quotient of the larger in the smaller is the constant wherewith to multiply the tooth values at large end, to obtain those at small end. In our example we find:

$$a \ l = 3.8 \\ a \ b = 5.8 = .655 = \text{constant}$$
 For 5 P we have:  
 $t = .3141$   $t' = .2057$   
 $s = .2000$   $s' = .1310$   
 $f = .0314$   $f = .0314$   
 $s + f = .2314$   $s' + f = .1624$   
 $D'' + f = .4314$ .  $s' = .1310$   
 $D''' + f = .2934$ 

From the foregoing it is evident that a spur gear cutter could not be used, since a bevel gear cutter must be thinner.

If in gears of more than 30 teeth the faces are proportionately long, we select a cutter whose curve corresponds to the midway section of the tooth. The curve of the cutter is found by the method explained in Part I. of this Chapter.

### CHAPTER IX.

# DIRECTIONS FOR CUTTING BEVEL GEARS WITH ROTARY CUTTER.

(Fig. 20.)

In order to obtain good results, the gear blanks must be of the right size and form. The following sizes for each end of the tooth must be given the workman:

Total depth of tooth.

Thickness of tooth at pitch line.

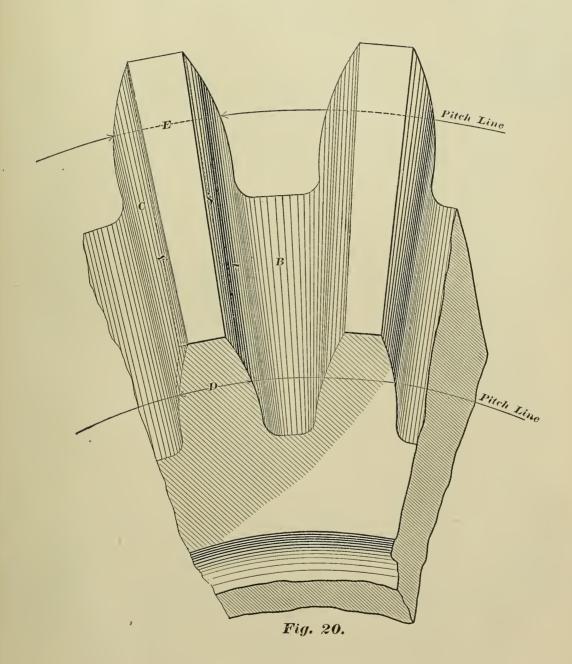
Height of tooth above pitch line.

These sizes are obtained as explained in Chapter VIII.

The workman must further know the cutting angle (see formula on page 13 and compare Chapter VIII.), and be provided with the proper tools with which to measure teeth, etc.

In cutting a gear on a universal milling machine the operations and adjustments of the machine are as follows:

- 1. Set spiral bed to zero line.
- 2. Set cutter central with spiral head spindle.
- 3. Set spiral head to the proper cutting angle.
- 4. Set the index on head for the number of teeth to be cut, leaving the sector on the straight or numbered row of holes, and set the pointer (or in some machines the dial) on cross-feed screw of milling machine to zero line.
- 5. As a matter of precaution, mark the depth to be cut for large and small end of tooth on their respective places.
- 6. Cut two or three teeth in blank to conform with these marks in depth. The teeth will now be too thick on both their pitch circles.
- 7. Set the cutter off the center by moving the saddle to or from the frame of the machine by means of the cross-feed screw, measuring the advance on dial of same. The saddle must not be moved further than what to good judgment



appears as not excessive; at the same time bearing in mind that an equal amount of stock is to be taken off each side of tooth.

- 8. Rotate the gear in the opposite direction from which the saddle is moved off the center, and trim the sides of teeth (A) (Fig. 20.)
- 9. Then move the saddle the same distance on the opposite side of center and rotate the gear an equal amount in the opposite direction and trim the other sides of teeth (C).
- ro. If the teeth are still too thick at large end E, move the saddle further off the center and repeat the operation, bearing in mind that the gear must be rotated and the saddle moved an equal amount each way from their respective zero settings.

It is generally necessary to file the sides of teeth above the pitch line more or less on the small ends of teeth, as indicated by dotted lines F F. This applies to pinions of less than 30 teeth.

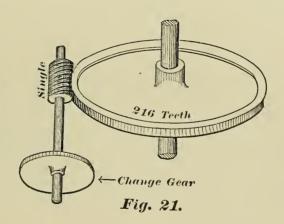
For gears of coarser pitch than 5 diametral it is best to make one cut around before attempting to obtain the tooth thickness.

The formulas for obtaining the dimensions and angles of gear blanks are given in Chapter III.

### CHAPTER X.

## THE INDEXING OF ANY WHOLE OR FRAC-TIONAL NUMBER.

(Fig. 21)



In indexing on a machine the question simply is: How many divisions of the machine index have to be advanced to advance a unit division of the number required. To which is the

$$answer = \frac{divisions of machine index}{number to be indexed}$$

Suppose the number of divisions in index wheel of machine to be 216.

EXAMPLE I.—Index 72.

Answer: 
$$\frac{216}{72} = 3$$
 (3 turns of worm).

EXAMPLE II.—Index 123.

$$\frac{216}{123} = 1 + \frac{93}{123}$$

If now we should put on worm shaft a change gear having 123 teeth, give the worm shaft, Fig. 21, one turn, and in addition thereto advance 93 teeth of the change gear (to give the fractional turn), we would have indexed correctly one unit of the given number, and so solved the problem. Should we not have change gear 123 we may try those on hand. The question then is: How many teeth  $(\chi)$  of the gear on hand (for instance 82) must we advance to obtain a result equal to the one when advancing 93 teeth of the 123 tooth gear? We have:

$$\frac{93}{123} = \frac{\chi}{82}$$
 where  $\chi = 62$ 

Example III.—Index 365, change gear 147.

$$\frac{216}{365} = \frac{\chi}{147}$$
 where  $\chi = 87 - \frac{3}{365}$ 

Here 147 is the change gear on hand. In indexing for a unit of 365 we advance87 teeth of our 147 tooth gear. It is evident that in so doing we advance too fast and will have indexed three teeth of our change gear too many when the circle is completed. To avoid having this error show in its total amount between the last and the first division, we can distribute the error by dropping one tooth at a time at three even intervals.

Example IV.—Index 190.

$$\frac{216}{190} = 1 + \frac{26}{190}$$
 Change gear on hand 88 T  
 $\frac{26}{190} = \frac{\chi}{88}$  where  $\chi = 12 + \frac{8}{190}$ 

To distribute the error in this case we advance one additional tooth ot a time of the change gear at eight even intervals.

Example V.—Index 117.3913.

$$\frac{216}{117.3913} = 1 + \frac{986087}{1173913}$$

This example is in nowise different from the preceding ones, except that the fraction is expressed in large numbers. This fraction we can reduce to lower approximate values, which for practical purposes are accurate enough. This is done by the method of continued fractions. [For an explana-

tion of this method we refer to our "Practical Treatise on Gearing."]

Note.—Find the first two fractions by reduction  $\frac{1}{1} = \frac{1}{1}$  and  $\frac{1}{1+\frac{1}{5}} = \frac{5}{6}$ ; the

others are then found by the rule  $\begin{cases} b \ c + a = d \\ b^1 \ c + a^1 = d^1 \end{cases}$ 

The fraction  $\frac{2}{2}\frac{1}{5}$  is a good approximation; putting therefore a change gear of 25 teeth on worm shaft, we advance (beside the one full turn) 21 teeth to index our unit.

Of course, in using any but the correct fraction we have an error every time we index a division; so that when indexed around the whole circle, we have multiplied this error by the number of divisions.

In the present example this error is evidently equal to the difference between the correct and the approximate fraction used. Reducing both common fractions to decimal fractions we have:

$$\frac{986087}{1173913} = .84000006$$

$$\frac{21}{25} = \frac{.84000000}{.00000006} = \text{error in each division.}$$

217

.00000006  $\times$  117.3913 = .00000704348 total error in complete circle. This error is expressed in parts of a unit division. (To find this error expressed in inches, multiply it by the distance between two divisions, measured on the circle.) In this case the approximate fraction being smaller than the correct one, in indexing the whole circle we fall short .00000704348 of a division.

Example VI.—Index 15.708

$$\frac{216}{15.708} = 13 + \frac{11796}{15708}$$

$$\frac{11796}{15708} = \frac{983}{1309}$$

$$983) \frac{1309}{326} = \frac{983}{326}$$

$$\frac{983}{326} = \frac{983}{5}$$

$$\frac{30}{26}$$

$$\frac{25}{25}$$

$$1) 5 (5$$

$$\frac{5}{5}$$

$$\frac{983}{1309} = \frac{1}{1+\frac{1}{3+\frac{1}{5}}}$$

$$\frac{1}{65+\frac{1}{5}}$$

$$\frac{1}{1} = \frac{3}{4} = \frac{196}{261} = \frac{983}{1309}$$

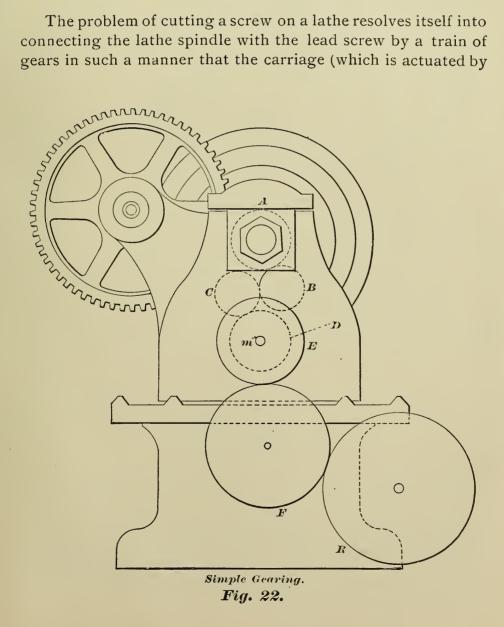
In using the approximation  $\frac{196}{261}$  the error for each division (found as above) will be .00002927, for the whole circle .0000460. In this case, the approximation being larger than the correct fraction, we overreach the circle by the error.

### CHAPTER XI.

### THE GEARING OF LATHES FOR SCREW CUTTING.

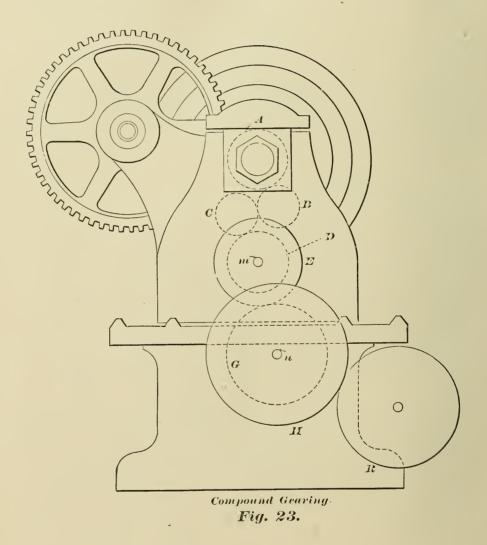
(Figs. 22, 23.)

The problem of cutting a screw on a lathe resolves itself into connecting the lathe spindle with the lead screw by a train of gears in such a manner that the carriage (which is actuated by



the lead screw) advances just one inch, or some definite distance, while the lathe spindle makes a number of revolutions equal to the number of threads to be cut per inch.

The lead screw has, with the exception of a very few cases, always a single thread, and to advance the carriage one inch it therefore makes a number of revolutions equal to its number



of threads per inch. Should the lead screw have double thread, it will, to accomplish the same result, make a number of revolutions equal to half its number of threads per inch. It follows that we must know in the first place the number of threads per inch on lead screw.

It ought to be clearly understood that one or more intermediate gears, which simply transmit the motion received from one gear to another, in no wise alter the ultimate ratio of a train of gearing. An even number of intermediate gears simply change the direction of rotation, an odd number do not alter it.

The gearing of a lathe to solve a problem in screw cutting can be accomplished by

A. Simple gearing.

B. Compound gearing.

Referring to the diagrams, Figs. 22 and 23, we have in Fig. 22 a case of simple, and in Fig. 23 a case of compound gearing.

In simple gearing the motion from gear E is transmitted either directly to gear R on lead screw or through the intermediate F. In compound gearing the motion of E is transmitted through two gears (G and H) keyed together, revolving on the same stud n, by which we can change the velocity ratio of the motion while transmitting it from E to R. With these four variables E, G, H, R, we are enabled to have a wider range of changes than in simple gearing.

B and C, being intermediate gears, are not to be considered. If, as is generally the case, gear A equals gear D, we disregard them both, simply remembering that gear E (being fast on same shaft with D) makes as many revolutions as the spindle. Sometimes gear D is twice as large as gear A, then, still considering gear E as making as many revolutions as the spindle, we deal with the lead screw as having twice as many threads per inch as it measures.

### SIMPLE GEARING.

Let there be: the number of teeth in the different gears expressed by their respective letters, as per Fig. 22, and

s = threads per inch to be cut, L = threads per inch on lead screw; then

If now one of the two gears E and R is selected, the other will be:

$$R = \frac{s}{L} \frac{E}{L}$$
;  $E = \frac{LR}{s}$ 

2. The two gears may be found by making

$$R = p s$$
  
 $E = p L$  where  $p$  may be any number.

3. The above holds good when a fractional thread is to be cut, but if the fraction is expressed in large numbers, as, for instance, s = 2.833 ( $2\frac{833}{1000}$ ), we first reduce this fraction ( $\frac{833}{1000}$ ) to lower approximate values by the process of continued fraction (see pages 73 and 74).

833) 
$$\frac{1000}{167}$$
 (1  $\frac{833}{167}$ ) 833 (4  $\frac{833}{165}$ ) 167 (1  $\frac{165}{2}$ ) 165 (82  $\frac{16}{5}$ )  $\frac{16}{5}$  (2)  $\frac{16}{5}$  (82  $\frac{16}{5}$ )  $\frac{4}{1}$ ) 2 (2  $\frac{2}{0}$ )  $\frac{1}{1}$   $\frac{4}{5}$   $\frac{1}{5}$   $\frac{4}{6}$   $\frac{414}{497}$   $\frac{833}{1000}$   $\frac{5}{0}$  = .833 (nearly) and  $s = 2\frac{5}{6}$ 

If in this case L = 4, and we select E = 48, then, since

$$R = \frac{s E}{L} \quad R = 34$$

### COMPOUND GEARING.

4. In a lathe geared compound for cutting a screw the product of the drivers (E and H. Fig. 23) multiplied by the number of threads per inch to be cut must equal the product of the driven (G and I) multiplied by the number of threads on lead screw. This is expressed by

E. H. 
$$s = G.R.L$$
 or  $\frac{E.H.s}{G.R.L} = r$ 

If three of the gears E, H, G, R have been selected, the fourth one would be either

$$E = \frac{G R L}{H s} \quad \text{or}$$

$$H = \frac{G R L}{E s} \quad \text{or}$$

$$G = \frac{E H s}{R L} \quad \text{or}$$

$$R = \frac{E H s}{G L}$$

$$s = \frac{R G L}{E H} = L \left(\frac{R \cdot G}{L \cdot E \cdot H}\right)$$

If a fractional thread is to be cut, as under "3," we reduce the fraction to lower approximate values.

Example.—Gear for 5.2327 threads per inch, lead screw is 6 threads.

$$.2327 = \frac{2327}{10000}$$

$$\frac{2327) \frac{10000}{692} (4) \frac{9308}{692} (2) \frac{2327}{2510} (3) \frac{2076}{2510} (3) \frac{190}{610} (3) \frac{183}{7} \frac{1}{610} (8) \frac{56}{5} \frac{5}{7} \frac{7}{15} \frac{1}{5} \frac{5}{27.5} \frac{1}{1000} \frac{2}{1000} \frac{1}{1000} \frac{37}{1000} \frac{37}{1000} \frac{306}{1000} \frac{343}{10000} \frac{343}{10000} \frac{37}{10000} \frac{306}{10000} \frac{343}{10000} \frac{343}{10000} \frac{37}{10000} \frac{306}{10000} \frac{343}{10000} \frac{343}{10000} \frac{37}{10000}  \frac{37}{10000}  \frac{37}{10000}  \frac{37}{10000} \frac{37}{10000} \frac{37}{10000} \frac{37}{10000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{100000} \frac{37}{1000000} \frac{37}{100000} \frac{37}{100000} \frac{37}{1000000} \frac{37}{1000000} \frac{37}{1000000} \frac{37}{1000000} \frac{37}{10000000} \frac{37}{10000000} \frac{37}$$

5. The examples so far given all deal with single thread. The pitch of a screw is the distance from center of one thread to the center of the next. The lead of a screw is the advance for each complete revolution. In a single thread screw the pitch is equal to the lead, while in a double thread screw the pitch is equal to one-half the lead; in a triple thread screw equal to one-third the lead, etc.

If we have to gear a lathe for a many-threaded screw (double, triple, quadruple, etc.), we simply ascertain the lead, and deal with the lead as we would with the pitch in a single thread screw, *i. e.*, we divide one inch by it, to obtain the number of threads for which we have to gear our lathe.

Example.—Gear for double thread screw, lead = .4654. Number of threads per inch to be geared for is:

$$\frac{1}{\text{Lead}} = \frac{1}{.4654} = 2.1.187$$

Lead screw is four threads per inch.

As in previous examples, we reduce the fraction .1487 =  $\frac{1487}{10000}$  to lower approximate values by the process of continued fraction.

From the different values received in the usual way we select:

$$\frac{1}{74}$$
 = .1487 (nearly) and 2.1487 =  $2\frac{1}{74}$ 

We have therefore:

$$s = 2\frac{11}{14}$$

$$L = 4$$

$$\begin{cases} E = 74 \\ G = 30 \\ H = 40 \end{cases}$$

$$R = \frac{E \cdot H \cdot s}{G \cdot L} = \frac{74 \cdot 40 \cdot 2\frac{11}{14}}{30 \cdot 4} = 53$$

Note.--In using any but the original fraction we commit an error. This error can be found by reducing the approximate fraction used to a decimal fraction, and comparing it with the original fraction. In the above example the original fraction is

In cutting a multiple screw, after having cut one thread, the question arises how to move the thread tool the correct amount for cutting the next thread.

In cutting double, triple, etc., threads, if in simple or compound gearing the number of teeth in gear E is divisible by 2, 3, etc., we so divide the teeth; then leaving the carriage at rest we bring gear E out of mesh and move it forward one division, whereby the spindle will assume the correct position.

When E is not divisible we find how many turns (V) of gear R are made to each full turn of the spindle. Dividing this number by 2 for double, by 3 for triple thread, etc., we advance R so many turns and fractions of a turn, being careful to leave the spindle at rest.

For compound gearing:

$$V = \frac{E \cdot H}{G \cdot R}$$

When the gear D is twice as large as the gear A (as explained in fifth paragraph, page 78.) the formula would be

$$V = \frac{E. H.}{2 G. R.}$$

If in simple gearing both E and R are not divisible, one remedy would be to gear the lathe compound; or the face-plate may be accurately divided in two, three or more slots, and all that is then necessary is to move the dog from one slot to another, the carriage remaining stationary.













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